

## Stress Analysis of Composite Spur Gear

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### ABSTRACT

*This paper investigates the static stress characteristics of an involute composite spur gear system including bending stresses and contact stresses of gears in mesh and comparing it with the existing involute cast iron spur gear system. The aim is to replace the cast iron spur gear with Carbon fibre epoxy composite spur gear due to its high strength, low weight and damping characteristics. A pair of involute spur gear is modelled in a CAD system (PRO/ ENGINEER) and FEA is done by using finite element software ANSYS 13. The bending stresses in the tooth root and contact stresses were examined using a 3-D FEM model. The bending stress obtained by finite element analysis method is compared with bending stress obtained by Lewis equation and the contact stress obtained by finite element analysis method is compared with contact stress obtained by Hertzian equation.*

### 2. INTRODUCTION

Gearing is one of the most critical components in a mechanical power

transmission system, and in most industrial rotating machinery. Many high-performance power transmission applications (e.g., automotive and aerospace) require low weight [6], [15], [19]. The two primary failure modes of gears are, one by tooth breakage from excessive bending stress and other by surface pitting or wear from excessive contact stress [9]. The conventional spur gears are continuously being investigated in order to reduce the failure or increase their transmissible power level, either by developing new composite materials [6], [7], [11] or by modifying the gear tooth geometry [16], [19], [20], [21], [22]. Carbon fibre epoxy has high strength and less density compared to cast iron and steel [7], [11], [15]. Short carbon fiber reinforced epoxy gear fabricated by properly controlled injection molding processes can provide higher strength and better performance and often exhibit less wear rate [10].

### 3. MODELLING OF SPUR GEAR

The Driving and driven gear are the most important components of the Gear box

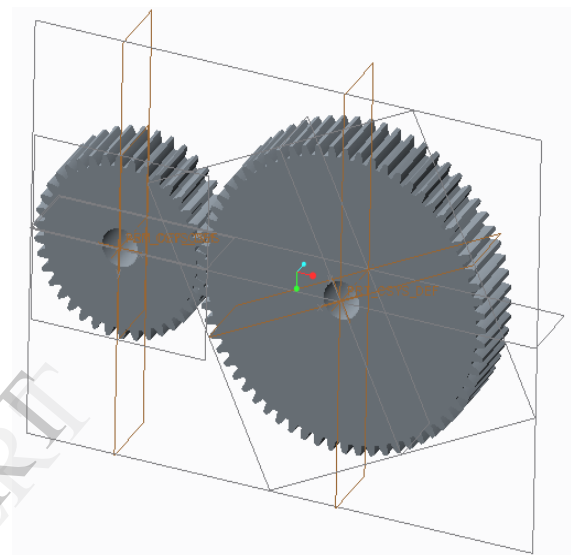
of any automotive. Modeling allows the design engineer to let the characteristic parameters of a product drive the design of that product. During the gear design, the main parameters that would describe the designed gear such as module, pressure angle, root radius, and tooth thickness, number of teeth could be used as the parameters to define the gear.

### Gear geometry

Parameter	Driving Gear	Driven Gear
Profile	Involute	Involute
Module (mm)	3	3
Pressure angle (deg)	20	20
No. of teeth	38	65
PCD(mm)	114	195
Addendum circle diameter	120	201
Base circle diameter	106.8	187.8
Root circle Diameter(mm)	106.5	187.5
Center Distance(mm)	154.5	154.5
Face Width(mm)	42	42
Tooth thickness(mm)	4.7124	4.7124

The finite element method is proficient to supply this information but the time required to generate proper model is a large. Therefore to reduce the modelling

time a pre-processor method that builds up the geometry required for finite element analysis may be used, such as Pro/Engineer. Pro/Engineer can generate three dimensional models of gears. The generated model geometry in Pro/Engineer is opened in ANSYS for analysis.



### 4. BENDING STRESS - LEWIS EQUATION

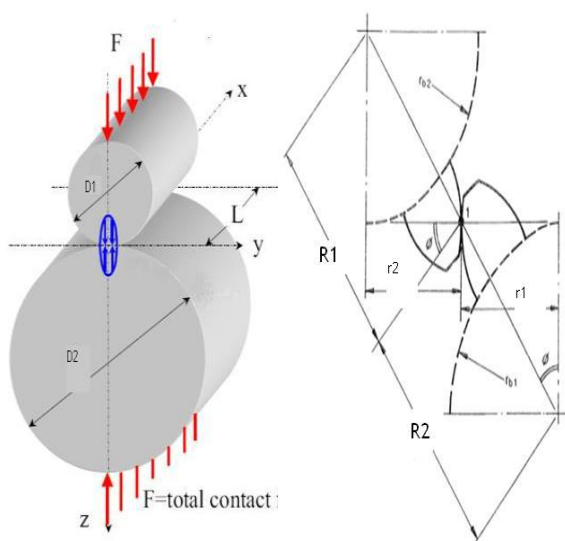
Bending failures in gears is predicted by comparing the calculated bending stress to experimentally-determined allowable fatigue values for the given material. This bending stress equation was derived by Wilfred Lewis [1], [2]. Lewis considered gear tooth as a cantilever beam with static normal force  $F$  applied at the tip of the tooth. Lewis equation to calculate bending stress is,

$$\sigma_b = \frac{F_t}{bYm}$$

Where,  $F_t$  = Tangential Force,  
 $b$  = Face width of tooth,  
 $Y$  = Lewis form factor  
 $m$  = Module.

## 5. CONTACT STRESS - HERTZIAN EQUATION

Contact failure in gears is currently predicted by comparing the calculated Hertz contact stress to experimentally determined allowable values for the given material. Hertz treated a pair of gear teeth as two cylinders of radii equal to the radii of curvature of the mating involutes at the pitch point [1], [2]. Contact stress is a compressive stress occurring at the point of maximum Hertzian stress.



The maximum contact stress/ Hertz stress/ Compressive stress/ Contact pressure,

$$P_{\max} = \sigma_H = \frac{4F}{\pi BL}$$

Where, Contact width

$$B = \sqrt{\frac{8 \times F}{\pi \times L} \times \frac{\frac{1-\nu_1^2}{E_1} + \frac{1-\nu_2^2}{E_2}}{\frac{1}{D_1} + \frac{1}{D_2}}}$$

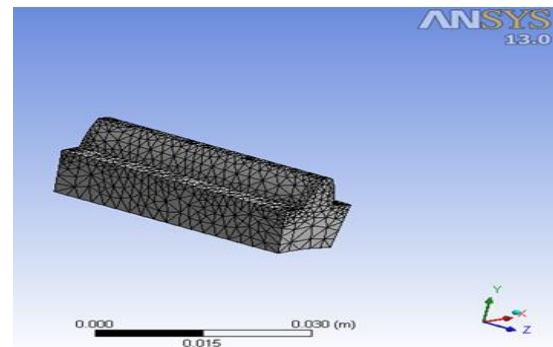
Where,  $F$  = Applied Force

$D_1$  &  $D_2$  = Diameters of the gears,

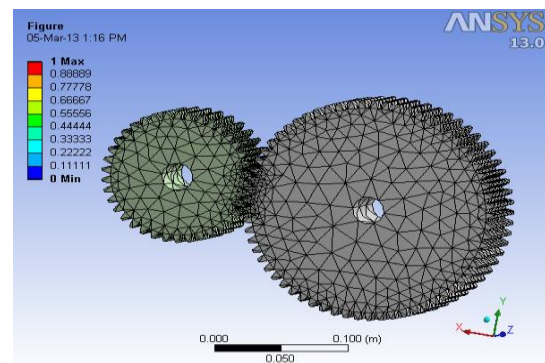
$E_1$  &  $E_2$  = Moduli of Elasticity of gear materials,

$\nu_1$  &  $\nu_2$  = Poisson's ratios of gear materials.

## 6. MESH GENERATED FOR FEA



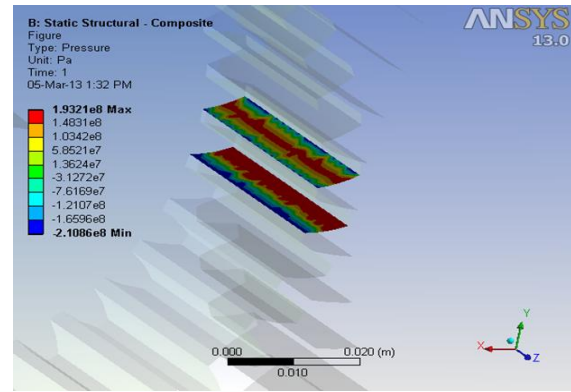
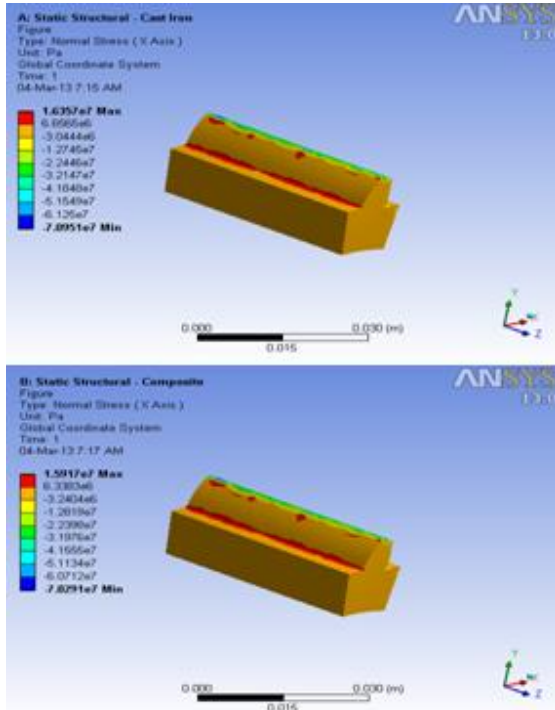
Meshing for bending stress analysis



Meshing for contact stress analysis

## 7. RESULTS

### 7.1. BENDING STRESS ANALYSIS



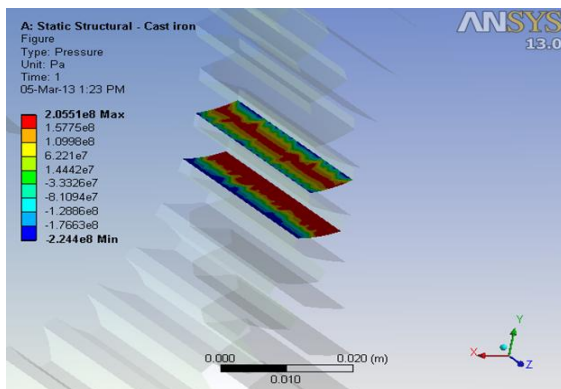
Material	Maximum Stress Induced (MPa)	
	Analytical Procedure	FEM Numerical Procedure
Cast Iron	200	205.51
Carbon fibre epoxy composite	188.51	193.21

Material	Maximum Stress Induced (MPa)	
	Analytical Procedure	FEM Numerical Procedure
Cast Iron	15.27	16.36
Carbon fibre epoxy composite	15.27	15.92

### 8. CONCLUSION

The objective of current work is to replace the cast iron spur gear with carbon fibre epoxy composite spur gear. For that, analytical and finite element method are applied for determining bending stresses and contact stresses of gear tooth. The obtained FEA results is compared with the analytical results and found that both results are comparable. Result shows that by both stress analysis the strength of the carbon fiber reinforced epoxy spur gear made by 90 degree fibre orientation of laminates is more when compared with cast iron spur gear . Also the density of the carbon fiber reinforced epoxy is very less when

### 7.2. CONTACT STRESS ANALYSIS



compared with cast iron spur gear. So we can conclude that the cast iron spur gear can be replaced by Carbon fiber reinforced epoxy (composite) spur gear due to its high strength, low weight and damping characteristics.

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