

# Spur Gear Designing and Weight Optimization

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**Abstract :-** In this paper, we are designing and optimizing of gear for event SAE BAJA Transmission. The given values of forces, torque and factor of safety of gear are calculated in accordance to custom vehicle. The gear is designed using SOLIDWORKS Toolbox. Gear weight is optimized by material removal from specific region. After it, simulation on SOLIDWORKS and ANSYS is done giving input values from calculation. The result shows the difference in weight and factor of safety between optimized and un-optimized gear models.

## INTRODUCTION :-

Gear is used gain mechanical advantage in power transmission. Gears of different types: Spur Gear, Helical Gear, Bevel Gear, Worm and Wheel Gear. Gears are commonly used in Transmission of Automobiles via Gearbox. Transmission assembly are stated: Simple Gear train, Compound Gear train, Planetary, etc. Weight of gears in gearbox becomes a constant factor of vehicle weight as whole. In college projects like SAE BAJA, SUPERA, etc. the basic knowledge of transmission is seen by an engineer skill to design a gearbox of required strength and maintain lightweight. The vehicle this gear is designed for uses a Engine power source with CVT.

## OBJECTIVE:-

- To choose material of high strength.
- To calculate the theoretical forces on spur gear by conventional formulas.
- To model the spur gears on SOLIDWORKS using toolbox spur gear.
- To remove material and optimize weight of spur gear.
- To simulate gear on SOLIDWORKS and ANSYS.

## ANALYSIS:-

### THEORETICAL ANALYSIS

#### Gear Material

Material for Gear is chosen based on yield strength of material. Materials commonly used are AISI 4340, EN 353, etc. We have taken AISI 4340 for its high strength. The properties are as in Table 1.

Table 1  
AISI 4340 Properties

Density	7.8gm/cm <sup>3</sup>
Hardness(BH)	320
Young's modulus	200GPa
Poisson's ratio	0.30
Yield strength	710MPa
Ultimate tensile strength	1110MPa

#### Engine

Engine used here is Briggs and Stratton engine, Model 19(Fig. 1). Its specifications are given in Table 2.

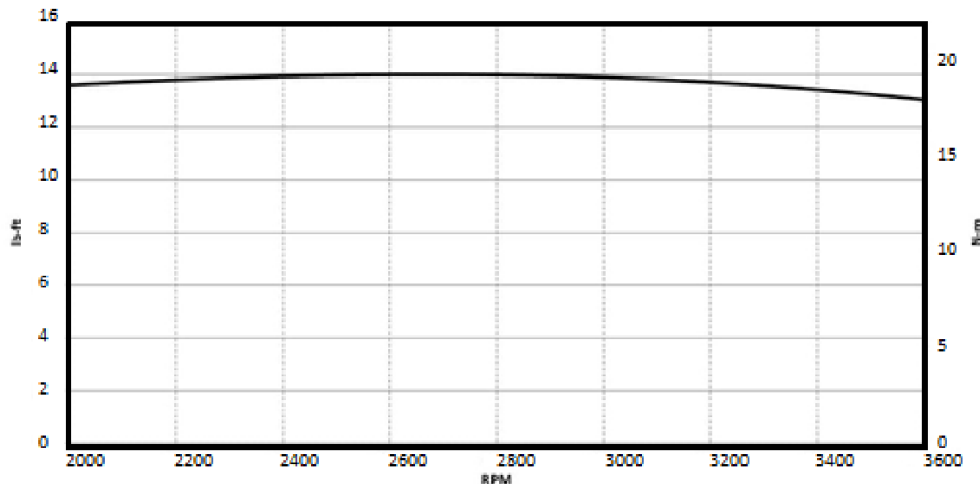


Fig. 1 B&S Power Graph

Table 2  
 Input Engine

Model	19L232-0054 G1
Compression ratio	8.1 to 1
Bore/Stroke	3.12" / 2.44"
HP(Gross)	10.0 Hp
Max. RPM	3800 rpm
Max. Torque	19.6 Nm at 2800 rpm

Continuous Variable Transmission (CVT) is used as constant reduction between engine output and gearbox input. Specifications are given in Table 3.

Table 3  
 CVT Specifications

CVT	Polaris
Max. ratio	3:1
Min. ratio	0.7:1

**Calculation**

**Torque on gear**

Torque is transferred from engine to gearbox after reduction from CVT. The final torque on gear is:-

$M_t$  – torque input at gear

$M_t = \text{max. torque of engine} * \text{max. CVT ratio} * \text{Gear reduction ratio}$

$$M_t = 19.6 * 3 * 7.54$$

$$M_t = 443 \approx 450 \text{ Nm}$$

**Spur Gear nomenclature**

Basic dimensions of Gear are given:-

Module(m)=2.5

Number of Teeth(z)=59

Pitch Circle Diameter(d)=  $m * z = 147.5 \text{ mm}$

Width of Gear(b)=20mm

Output Shaft Diameter=36mm

**Calculation of Tangential Force**

The two components of net force on gear tooth are: Radial force( $F_r$ ), Tangential force( $F_t$ ). The only force responsible of gear rotation is  $F_t$ , calculated below:-

$F_t$  – Tangential force applied on gear tooth/teeth

$$F_t = 2 * M_t / d \tag{1}$$

$$F_t = 2 * 450 * 1000 / 147.5$$

$$F_t = 6101.6949 \text{ N}$$

**Calculation of Beam Strength of Gear Teeth**

The resistive force offered by gear tooth is calculated below:-

$F_{en}$  – Resistive force of Gear tooth

$\sigma_{en}$  – Allowable permissible stress of material

$\sigma_{ut}$  – Ultimate tensile stress of material

$$\sigma_{en} = \sigma_{ut} / 3 \tag{2}$$

$$F_{en} = \sigma_{en} * b * Y * m$$

$$F_{en} = (1110/3) * 20 * [\pi * (0.175 - 0.95 / 59)] * 2.5$$

$$F_{en} = 9235.0843 \text{ N}$$

**Calculated Factor of Safety(FOS)**

Factor of Safety is calculated:-

$$FOS = F_{en} / F_t$$

$$FOS = 9235.0843 / 6101.6949$$

$$FOS = 1.51$$

**GEAR DESIGNING**

For Gear model, we take a standard spur gear from SOLIDWORKS Toolbox. Assign the Module and Teeth number. The software will generate a gear using Global Equations. We optimize it according to design using Static Simulation.

For simulation the center hub is fixed fixture and tangential force is applied on teeth for worst condition possible.

Simulation of Gear on SOLIDWORKS Fig. 2, 3.

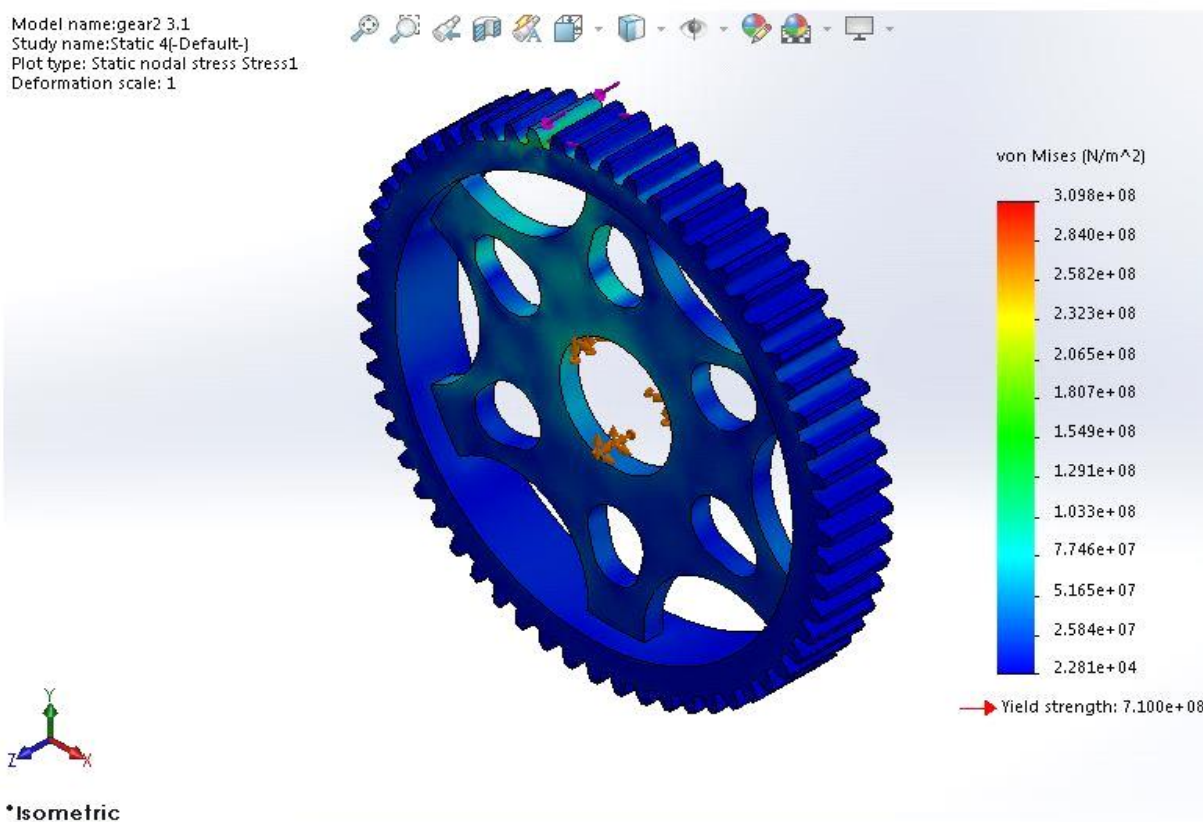


Fig. 2 Stress

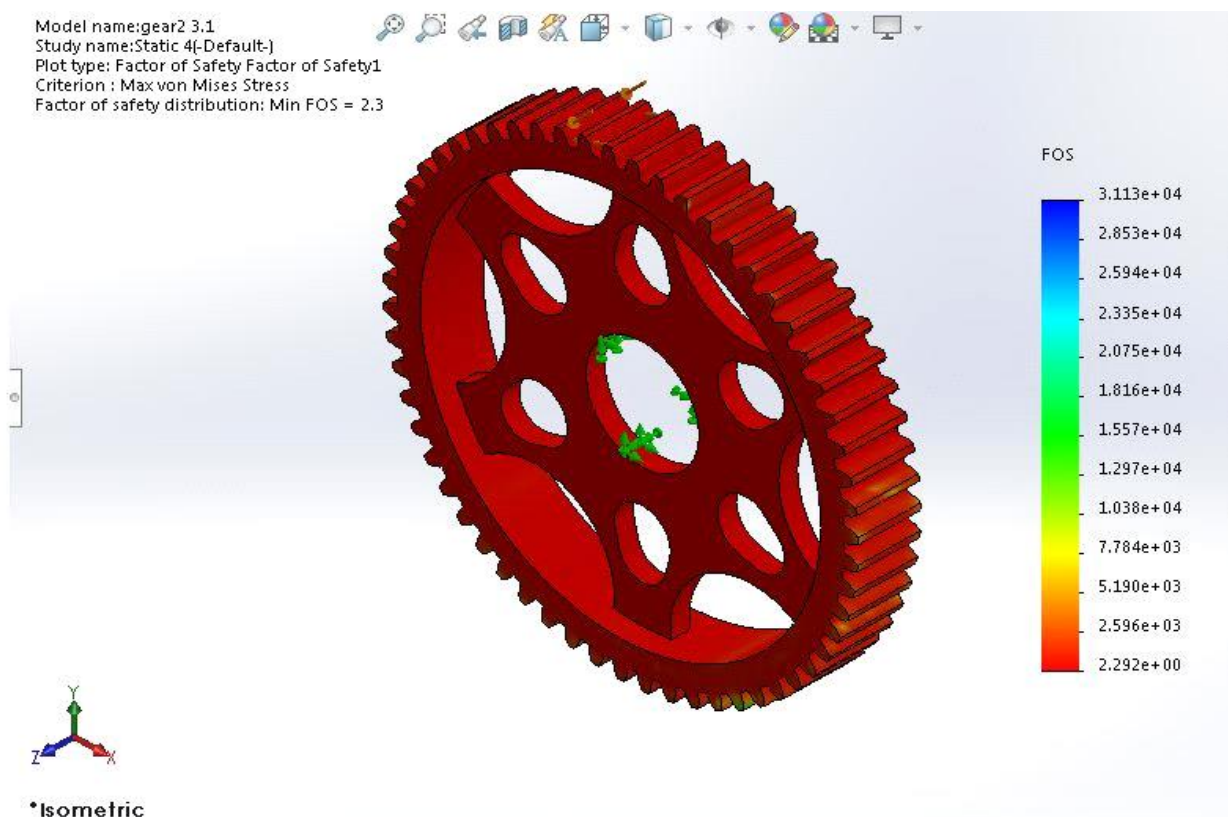


Fig. 3 Factor of Safety

Simulation of Gear on ANSYS Fig. 4, 5.

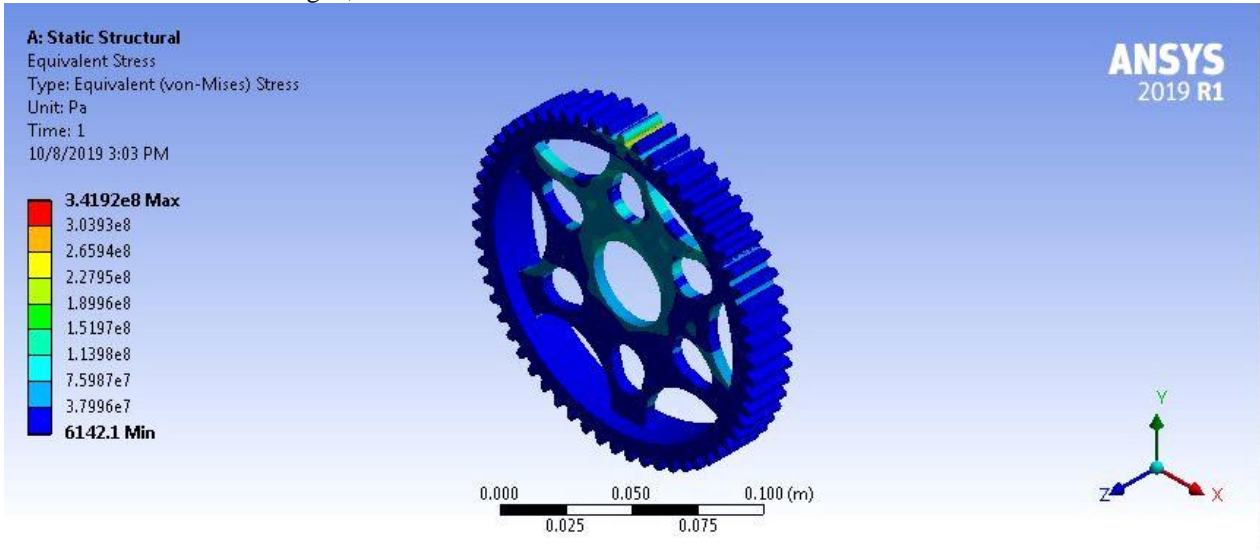


Fig. 4 Stress

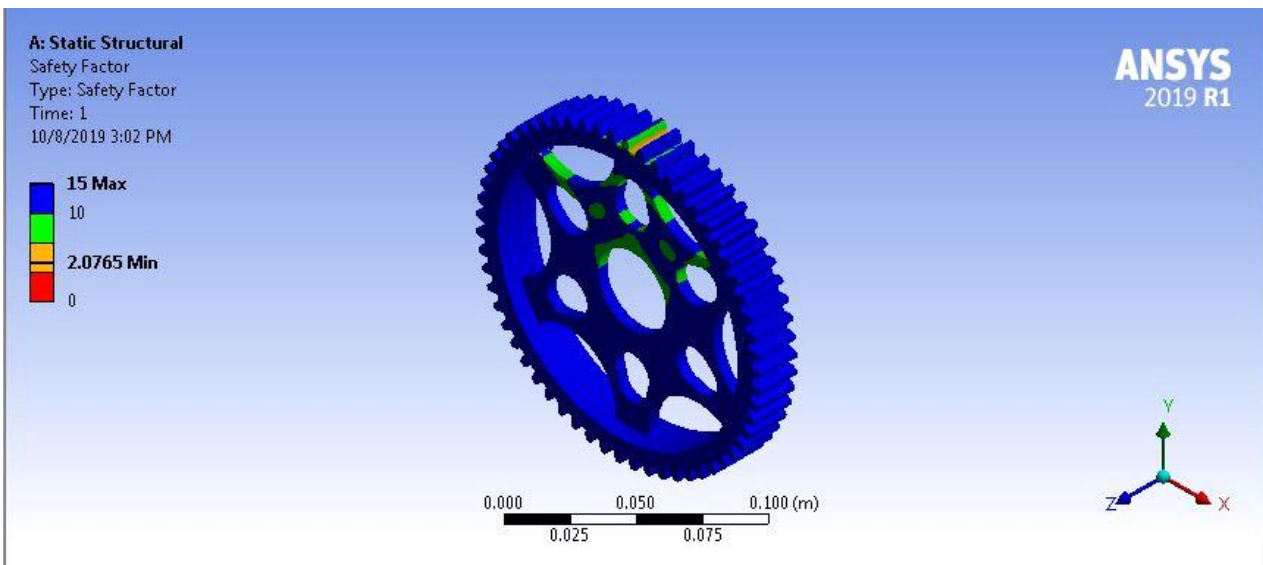
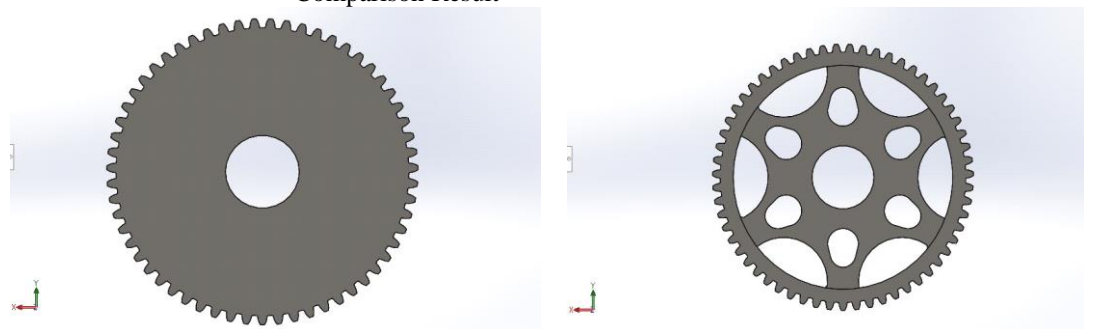


Fig. 5 Factor of Safety

**RESULT :-**

After simulations, we got the following results give in Table 4.

**Table 4**  
**Comparison Result**



Normal Gear	Optimized Gear
FOS=2.24	FOS=2.01
Mass=2505.51gm	Mass=888.86gm

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