Peristaltic Transport Of A Conducting Bingham Fluid In A Channel

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Abstract
Peristaltic transport of a conducting Bingham fluid in a channel under long wavelength and low Reynolds number assumptions is studied. This model is probably suitable for the blood flow in the sense that erythrocytes region and the plasma regions may be described as plug flow and non-plug flow regions. Further the blood is experimentally proved to be a conducting fluid in several physiological applications. Motivated by this fact, it is proposed to study the peristaltic transport of a Bingham fluid. It is observed that for a Bingham fluid the pressure difference and the mechanical efficiency of pumping depend on the yield stress and the magnetic field.

1. Introduction
Peristaltic pumping is a form of fluid transport generally from a region of lower to higher pressure, by means of a progressive wave of area contraction or expansion, which propagates along the length of a tube-like structure. Peristalsis occurs naturally as a means of pumping physiological fluids from one place in the body to another. Some electro-chemical reactions are held responsible for this phenomenon. This mechanism occurs in swallowing of food through oesophagus, in the ureter, the gastrointestinal tract, the bile duct and even in small blood vessels.

2. Nomenclature

<table>
<thead>
<tr>
<th>Symbol</th>
<th>Description</th>
</tr>
</thead>
<tbody>
<tr>
<td>a</td>
<td>Half width of the wave</td>
</tr>
<tr>
<td>b</td>
<td>Amplitude of the wave</td>
</tr>
<tr>
<td>B₀</td>
<td>Applied magnetic field</td>
</tr>
<tr>
<td>c</td>
<td>Wave speed</td>
</tr>
<tr>
<td>E</td>
<td>Mechanical efficiency</td>
</tr>
<tr>
<td>F</td>
<td>Dimensionless friction force</td>
</tr>
<tr>
<td>h</td>
<td>Height of the wave in the direction of y</td>
</tr>
<tr>
<td>M</td>
<td>Hartmann number</td>
</tr>
<tr>
<td>p</td>
<td>Dimensionless pressure</td>
</tr>
<tr>
<td>ΔP</td>
<td>Pressure rise/drop</td>
</tr>
<tr>
<td>q</td>
<td>Volume flux</td>
</tr>
<tr>
<td>Q</td>
<td>Time averaged flow rate</td>
</tr>
<tr>
<td>Rₑ</td>
<td>Reynold’s number</td>
</tr>
<tr>
<td>t</td>
<td>Time</td>
</tr>
<tr>
<td>U,V</td>
<td>Axial and transverse velocity components in laboratory frame</td>
</tr>
<tr>
<td>u,v</td>
<td>Axial and transverse velocity components in wave frame</td>
</tr>
<tr>
<td>u₀</td>
<td>Velocity in plug flow region</td>
</tr>
<tr>
<td>x,y</td>
<td>Axial and transverse co ordinates</td>
</tr>
<tr>
<td>X,Y</td>
<td>Dimensionless axial and transverse co ordinates</td>
</tr>
<tr>
<td>y₀</td>
<td>Plug flow region</td>
</tr>
<tr>
<td>λ</td>
<td>Wave length</td>
</tr>
<tr>
<td>σₑ</td>
<td>Electrical conductivity</td>
</tr>
<tr>
<td>Ψ</td>
<td>Stream function</td>
</tr>
<tr>
<td>φ</td>
<td>Amplitude ratio (b/a)</td>
</tr>
<tr>
<td>τ₀</td>
<td>Yield stress</td>
</tr>
<tr>
<td>μ</td>
<td>Viscosity index</td>
</tr>
</tbody>
</table>

3. Mathematical formulation and solution
Consider the peristaltic pumping of a conducting Bingham fluid in a channel of half-width (a). A longitudinal train of progressive sinusoidal waves takes place on the upper and lower walls of the channel. For simplicity, we restrict our discussion to the half width of the channel as shown in figure 4.1. The region between Y = 0 and Y = y₀ is called plug flow region. In this region, |τₓ,y| ≤ τ₀. In the region between Y =
\[ Y_0 \text{ and } Y = H, \quad |\tau_{y,x}| > \tau_o. \]  The wall deformation is given by
\[
H(x, t) = a + b \sin \left( \frac{2\pi}{\lambda} (x - ct) \right) \quad (1)
\]

The non-dimensional form of the equations governing the motion (dropping the bars) is
\[
\frac{d}{dy} \left( \tau_o - \Psi_{yy} \right) + M^2 \Psi_y = -\frac{dp}{dx} \quad (3)
\]
\[
0 = \frac{\partial p}{\partial y} \quad (4)
\]

where \( M = \sqrt{\frac{B_o a^2 \sigma c}{\mu}} \), the Hartmann number

The non-dimensional boundary conditions are

\[ \psi = 0; \quad \psi_{yy} = \tau_o \quad \text{ at } y = 0 \]

\[ u = \psi_y = -1 \quad \text{ at } y = h \quad (5) \]

Where \( h = 1 + \varphi \sin 2\pi x \) \quad (6)

5. Solution

Solving equation (4) with the boundary conditions (5), we obtain the velocity as
\[
u = A_1 e^{My} + B_1 e^{-My} + \frac{p}{M^2} \quad (7)\]
where $A_1 = a_1 p + a_2$, $B_1 = a_3 p + a_4$

$$a_1 = \left[ -\frac{1}{M^2} + \frac{1}{M^2} e^{-Mh} \right]$$
$$a_2 = \frac{-1 - \tau_e e^{-Mh}}{2 \text{Sinh } M}$$
$$a_3 = \frac{-1 + \tau_e e^{-Mh}}{2 \text{Sinh } M}$$
$$a_4 = \frac{\tau_e e^{Mh} + 1}{2 \text{Sinh } M}$$

$p = -\frac{\partial p}{\partial x}$

Taking $y = y_0$ in equation we get the velocity in plug flow region as

$$u_p = A_1 e^{My_0} + B_1 e^{-My_0} + \frac{P}{M^2} (0 \leq y \leq y_0)$$

(9) the volume flux $q$ through each cross-section in the wave frame is given by

$$q = \int_0^{y_0} u_p dy + \int_{y_0}^y u dy = a_5 p + a_6$$

(10)

where $a_5 = a e^{My_0} \left( \frac{1}{M} - a_1 e^{My_0} \right)$

$$a_6 = a e^{My_0} \left( 1 - \frac{1}{M} \right) + a_4 e^{-My_0} \left( 1 + \frac{1}{M} \right) + a_1 e^{My_0} - a_4 e^{-My_0} - \frac{h}{M^2}$$

From the above equation

$$\frac{\partial p}{\partial x} = \frac{a_6 - q}{a_5}$$

(11)

Averaging equation over one period yields the time mean flow (time-averaged volume flow rates) $\overline{Q}$ as

$$\overline{Q} = \frac{1}{T} \int_0^T Q dt$$

$$= q + 1$$

(12)

6. The Pumping characteristics

Integrating the equation (10) with respect to $x$ over one cycle of the wave as

$$\Delta P = \int_0^1 \left( \frac{a_6 - \overline{Q} + 1}{a_5} \right) dx$$

(13)

The pressure rise required to produce zero average flow rate is denoted by $\Delta P_0$ and is given by

$$\Delta P_0 = \int_0^1 \left( \frac{a_6 + 1}{a_5} \right) dx$$

(14)

The dimensionless friction force $F$ at the wall across one wavelength is given by

$$F = \int_0^1 \left( \frac{-dp}{dz} \right) dx$$

(15)

7. Mechanical Efficiency:

The mechanical efficiency of pumping for one wave length is given by

$$E = \frac{\overline{Q} \Delta P}{\int_0^1 p_s (h-1) dx}$$

$$= \frac{\overline{Q} \left[ I_1 - \overline{Q} I_2 \right]}{Q \left[ I_3 - \overline{Q} I_4 \right]}$$

(16)

Where

$$I_1 = \int_0^1 \frac{a_6}{a_5} dx$$

$$I_2 = \int_0^1 \frac{1}{a_5} dx$$

$$I_3 = \int_0^1 \frac{a_1}{a_5} dx$$

$$I_4 = \int_0^1 \frac{1}{a_5} dx$$
\[I_3 = \int_{0}^{a_6} \frac{1}{a_5} \sin 2\pi x \, dx\]

\[I_4 = \int_{0}^{1} \frac{1}{a_5} \sin 2\pi x \, dx\]

8 Discussion of the Results.

From the equation 13 we have calculated the pressure difference as a function of \(\overline{Q}\) for different values of yield stress \(\tau_0\) for a fixed \(\phi\) and shown in Fig. 2. It is observed that for a given \(\Delta P\) the flux \(\overline{Q}\) depends on yield stress and it decreases with increase in yield stress. For a given flux \(\overline{Q}\) the pressure rise \(\Delta P\) decreases with increase in yield stress \(\tau_0\). For free pumping, the flux \(\overline{Q}\) increases with decreasing yield stress.

From the variation of pressure rise with time averaged flow rate is calculated from equation (13) for different amplitude ratios and shown in Fig. 3. For given \(\Delta P\) the flux \(\overline{Q}\) for Bingham fluid depends on \(\phi\) and we find that the larger the amplitude ratio the greater the pressure rise against which the pump works. This observed for small values of \(\Delta P\) \((\Delta P < 0001)\) and free pumping cases.

From equation (13) we have calculated the variation of pressure rise with time averaged flow rate \(\overline{Q}\) for different magnetic parameters \(M\) and is shown in Fig. 4 we observed that the larger the magnetic parameter the greater the pressure rise against which the pump works. For free pumping the flux \(\overline{Q}\) depends on magnetic field and it increases with increasing magnetic parameter \(M\).

From equation (15), we have calculated the frictional force as a function of \(\overline{Q}\) for fixed amplitude ratio, and is depicted in Fig. 5. It is observed that the frictional force \(F\) has the opposite behaviour compared to pressure rise \(\Delta P\). As \(\tau_0\) increases the frictional force also increases with the flux \(\overline{Q}\). For a given \(\overline{Q}\) the frictional force increases with decrease in yield stress.

From equation (16) we have calculated the mechanical efficiency as a function of \(\overline{Q}\) and is depicted in Fig. 6 and Fig. 7. for different values of magnetic parameters \(M\), yield stress and amplitude ratio \(\phi\). For the chosen parameter the mechanical efficiency is positive for \(\overline{Q}\) less than or equal to 6 and is negative for \(\overline{Q}\) greater than 6. Further we observed that the pumping efficiency is greater for higher values of magnetic parameters. It is also found that the mechanical efficiency decreases with the increasing yield stress. For lower value flux \(\overline{Q}\), the efficiency remains constant and is unaffected by variation in \(M\) and \(\tau_0\).

From equation (14), we have calculated the pressure rise required to produce zero average flow rate \(\Delta P_0\) as a function of amplitude ratio \(\phi\) for different values of yield stress \(\tau_0\) and is shown in Fig. 8. It is observed that for a conducting Bingham fluid, the value of \(\Delta P_0\) increases with decrease in yield stress \(\tau_0\). It is also observed that the increase in the value of \(\Delta P_0\) is more for the amplitude ratio beyond 0.7.

From equation (14), we have calculated the pressure rise required to produce zero average flow rate \(\Delta P_0\) as a function of amplitude ratio \(\phi\) for different values of magnetic parameter \(m\) and is shown in Fig. 9. It is observed that for a conducting
Bingham fluid, the value of $\Delta P_0$ increases with increase in magnetic parameter.

Fig. 2 The variation of $\Delta P$ with averaged flow rate $\bar{Q}$ for different $\tau_0$ with $\varphi = 0.6$, $m = 0.02$, $y_0 = 0.2$

Fig. 3 The variation of $\Delta P$ with averaged flow rate $\bar{Q}$ for different $\varphi$, with $\tau_0 = 0.01$, $m = 0.01$, $y_0 = 0.2$

Fig. 4 The variation of $\Delta P$ with averaged flow rate $\bar{Q}$ for different $m$ with $\tau_0 = 0.01$, $\varphi = 0.6$, $y_0 = 0.2$

Fig. 5 The variation of $\Delta F$ with averaged flow rate $\bar{Q}$ for different $\tau_0$ with $\varphi = 0.6$, $m = 0.02$, $y_0 = 0.2$
Fig. 6 The variation of Mechanical Efficiency $E$ with averaged flow rate $\bar{Q}$ for different $m$ with $\tau_0 = 0.01$, $\phi = 0.6$, $y_0 = 0.1$

Fig. 7 The variation of Mechanical Efficiency $E$ with averaged flow rate $\bar{Q}$ for different $\tau_0$ with $\phi = 0.6$, $m = 0.02$, $y_0 = 0.1$

Fig. 8 The variation of $\Delta P_0$ with Amplitude ratio $\phi$ for different $\tau_0$ with $m = 0.02$, $y_0 = 0.2$

Fig. 9 The variation of $\Delta P_0$ with Amplitude ratio $\phi$ for different $m$ with $\tau_0 = 0.01$, $y_0 = 0.2$

References


