

# Modern Mathematical Techniques for Analysis of Temperature Distribution in Straight Profile Fins.

Vishnu Agarwal<sup>1)</sup>, Gaurav Chaturvedi<sup>2)</sup>, Dr. Alka Agrawal<sup>3)</sup>

<sup>1)</sup> Ph. D, Research Scholar, Department of Renewable Energy Engineering, CTAE, Udaipur,

<sup>2)</sup> ME Research Scholar, Department of Mechanical Engineering, University Institute of Technology .RGPV, Bhopal (M.P),

<sup>3)</sup> Professor & Head, Department of Mechanical Engineering, University Institute of Technology, RGPV, Bhopal (M.P).

**Abstract** - A fin is an extended surface device attached to the surface of a structure protruding into the adjacent fluid, where its purpose is to increase the heat transfer between the solid surface and the fluid. Because of the wide ranging applications, the analysis of fin heat transfer is of great significance. The equations governing fins with temperature dependent conductivity are nonlinear diffusion type differential equations. Due to the mathematical complexity of these equations exact analytical solutions are not easily tractable. An approximate solution to the original problem can be obtained by using Numerical & Analytical methods or combination of both approaches. In the past, investigations were focused on analytical solutions because they lacked the advantages of today's powerful computers to develop numerical solutions. The present paper illustrates the advanced mathematical schemes for analyzing the temperature profile of the Conductive-Convective rectangular linear fin of straight profile which is the solution of second order differential equation. Linear Differential fin equations are solved through Bessel functions, which gave standard Exact solution then solved by Approximated method, Numerical methods either iterative or non- iterative i.e. Power Series Solution, Finite Difference Technique and modern methods like Differential Transformation Method (DTM) further comparison is made from results obtained by different method in tabular and graphical form. At the end, the paper recommends that method which is fast to calculate and also gives reliable as well as accurate result for calculation of linear fin.

**Keywords:** Heat Transfer Fin, Straight Profile, Bessel's Functions, Finite Difference, Power Series, DTM.

## 1. INTRODUCTION

The ability to dissipate heat effectively with minimum cost has always been an engineering concern. A paper by **Harper and Brown (1922)** appeared as an NACA report. Which provide a mathematical analysis of the interesting interplay between convection and conduction in single extended surfaces? Harper and Brown called it a *cooling fin*, which

has known merely as a *fin*. Fins are widely used to augment the rate of heat transfer from the primary surface to the ambient medium in a large variety of thermal equipment. Typical components are found in such diverse applications are air-land-space vehicles and their power sources, in chemical, refrigeration, and cryogenic processes, in electrical and electronic equipment, in convention furnaces and gas turbines, in process heat dissipaters and waste heat boilers, and in nuclear-fuel modules. The main objective is to delineate three computational procedures for solving the

descriptive equation for a rectangular fin of straight profile. The three computational procedures are the power series method, the finite-difference technique and the differential transformation method.

## Fin Selection

The fins are designed and manufactured in many forms. The simplest type of fins for the analysis and manufacturing is the longitudinal fins of rectangular straight profile or constant thickness.

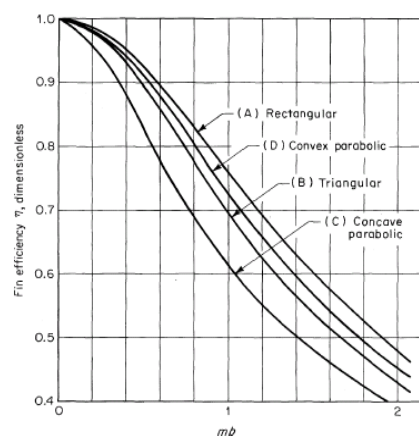


Figure 1 Graph for the efficiency of straight (Longitudinal) fins. The thermal design of constant fin thickness is also relatively simple. A straight rectangular fin is an extended surface attached to a plane wall. It may be uniform in cross sectional area, or its area may vary along its length to form a triangular, parabolic or trapezoidal shape. Therefore the study on extended surfaces begins with this. The problem for selection of fins of maximum efficiency (shown in figure), less volume and weight are also overcome by its usage.

## Modeling

The heat that is conducted through a body must frequently be removed by some convection process. So the analysis of combined conduction-convection system is very important from a practical standpoint.

Consider the one dimensional rectangular fin exposed to surrounding fluid at temperature  $T_{\infty}$  and at Tip of fin  $T_0 = 0$ ,

as shown in figure, Assuming heat flow is unidirectional along b.

Energy In (left face) = Energy out (Conduction through material) + Energy out (by convection to surrounding).

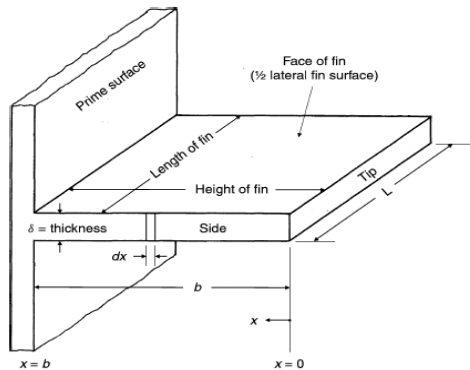


Figure 2: Terminology for the Rectangular fin of straight profile

$$\frac{d^2\theta}{dx^2} - \frac{hP}{KA_C}\theta = 0$$

$$\frac{d^2\theta}{dx^2} - M^2\theta = 0 \tag{1}$$

Where  $\frac{hP}{KA_C} = M^2$

$\frac{T - T_\infty}{T_o - T_\infty}$  (Dimensionless temperature)

Boundary conditions

$$X=0 \quad \frac{d\theta}{dX} = 0 \quad ;$$

$$X=1 \quad \theta=1 \quad ;$$

By solving differential equation (1) and applying boundary conditions

$$\theta = \theta_0 \frac{\cosh m(b-x)}{\cosh mL} \tag{2}$$

**Mathematical Analysis**

The mathematical schemes for obtaining solution of the temperature distribution along the length of Heat transfer Fins which is Conductive-Convective type are as follows.

- i) Exact Analytical Solution.
- ii) Approximate analytical solution.
- iii) Numerical solution schemes.

These solutions are based on the assumption that the Thermal Conductivity (K) and heat transfer coefficient (h) are constant throughout the length of the fin.

**Exact Analytical solution**

The profile of the fin is governed in the form of second order differential equations which via a transformation can be converted into the Bessel Equation and compared with generalized form of Bessel's equation. Then solved it using the Bessel functions (Theoretical Method) which give standard exact solution. This method found to be important

because it represent the solutions to the many differential equations.

(1)	(2)	(3)	(4)	(5)
X	M <sup>2</sup> =1	M <sup>2</sup> =1.5	M <sup>2</sup> =2.0	M <sup>2</sup> =6.572
0	0.648054	0.540981	0.459185	0.153145
0.2	0.661059	0.55729	0.47767	0.17372
0.4	0.700594	0.607202	0.534612	0.240971
0.6	0.768246	0.693726	0.634595	0.372969
0.8	0.86673	0.822079	0.78567	0.605181
1	1	1	1	1

Table 1- Temperature distribution by theoretical method

**Finite Difference Techniques**

To utilize this numerical method, equation of finite approximation written for each node within the material and resultant system of equation solved for the temperature at the various nodes.

Fin region  $0 \leq X \leq 1$  is derived into I equal intervals of size

$$\Delta X = \frac{1-0}{I}$$

,resulting in I+1 nodes for i=0,1,2,3.....I as

in figure.

Derivative for II order equation can be written as

$$\frac{d^2\theta}{dR^2} \Big|_i = \frac{\theta_{i+1} - 2\theta_i + \theta_{i-1}}{(\Delta X)^2}$$

Truncation error is of order  $(\Delta X)^2$

Substitute the values in differential equation

$$\frac{\theta_{i+1} - 2\theta_i + \theta_{i-1}}{(\Delta X)^2} - M^2\theta_i = 0$$

this equation is equation generator which gives a system of i algebraic equations for node temperatures  $\theta_i$ , where i=1,2,3.....I. Two additional equations are needed to equalize number

of equation in system of equation to the number of node temperature ( $\theta_i$ ).these two relations come from two boundary conditions.

(a) At Fin base (i=0)  $\theta_0=1$ , at X=1

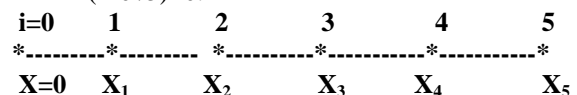
(b) At Fin tip (i=I)  $\frac{d\theta}{dX} = 0$  at X=0

Incorporate these Boundary Conditions in finite difference form  $\theta_0=1$  at last node i=I, derivative represented by two

point central formulation  $\frac{d^2\theta}{dR^2} \Big|_I = \frac{\theta_{i+1} + \theta_{i-1}}{2\Delta X}$  with

truncation error of  $(\Delta X)^2$  Set off equality  $\theta_{I+1} = \theta_{I-1}$  and put in equation, Convective fin of rectangular profile I divided into four equal interval of size

$$\Delta X = (1-0/5)=0.2$$



From equation generator three nodes temperature  $\theta_1, \theta_2, \theta_3$  regulated by following algebraic equation.

$$i=1 : \theta_{i+1} - [2 + M^2(\Delta X)]\theta_1 + \theta_{i-1} = 0$$

We obtained five equation of temperature profile which is representing here in matrix form.

$$[A] [\theta] = [S]$$

By using MATLAB solution of this matrix for the temperature profile  $\theta$  at different location of the fin from base (X=1) to Tip (X=0) at interval of 0.2. By taking the various values of constant  $M^2$  and solved by MATLAB we get solution which are represent here in tabular form.

		$M^2=6.452$	$M^2=2.0$	$M^2=1.5$	$M^2=1.0$
S no	X	FDM	FDM	FDM	FDM
1	0.0	0.1572	0.4610	0.5423	0.6489
2	0.2	0.1779	0.4784	0.5586	0.6619
3	0.4	0.2454	0.5362	0.6084	0.7013
4	0.6	0.3774	0.6354	0.6947	0.7688
5	0.8	0.6087	0.7865	0.8227	0.8671
6	1.0	1	1	1	1

Table 2 - Temperature distribution by Finite difference

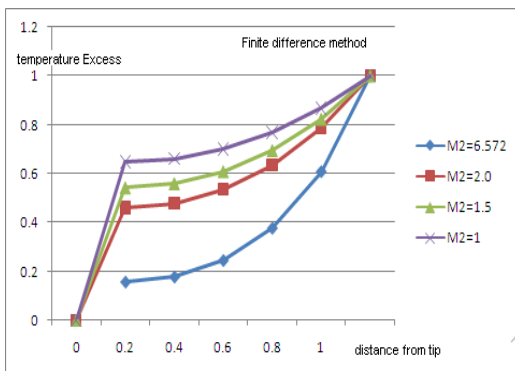


Figure-3: Temperature excess at various values of  $M^2$

Clearly the larger the number of nodes, the more complex and time consuming the solution, even with high speed computer. Several Software packages are available for solution of simultaneous equations, including MathCAD, TK solver, Mat Lab and Microsoft Excel.

**Power Series Method**

The method is a robust analytical method for solving linear differential equations, Let variable coefficient second order linear homogeneous equation is

$$a_o(x)y'' + a_1(x)y' + a_2(x)y = 0$$

$$y'' + \frac{a_1(x)}{a_o(x)}y' + \frac{a_2(x)}{a_o(x)}y = 0 \quad ; \text{Where } a_o(x) \neq 0$$

$$y'' + p(x)y' + q(x)y = 0 \quad (3)$$

in this equation the variable coefficient are most likely polynomials represented by power series. A solution of this equation in terms of power series form

$$y(x) = c_0 + c_1(x-x_0) + c_2(x-x_0)^2 + \dots = \sum_{m=0}^{\infty} c_m(x-x_0)^m \quad \dots (4)$$

Where the constants  $c_1, c_2, c_3, c_4, \dots$  are the coefficient and  $x_0$  is the center of power series.

$$\frac{dy}{dx} = \sum_{m=1}^{\infty} m c_m (x-x_0)^{m-1}$$

$$\frac{d^2y}{dx^2} = \sum_{m=2}^{\infty} m(m-1)c_m(x-x_0)^{m-2}$$

Substitute values of  $y, \frac{dy}{dx}$  and  $\frac{d^2y}{dx^2}$  in differential equation. Then equate the equation to zero the co-efficient of lowest power of x, this gives a quadratic equation in m which is known as **indicial equation**. Equate to zero the coefficients of other powers of x to find  $c_1, c_2, c_3, \dots$  in terms of  $c_0$ . Substitutes the values of constants in equation (4) to get the series solution of the equation (3). It is the iterative method which is used truncation of the series.

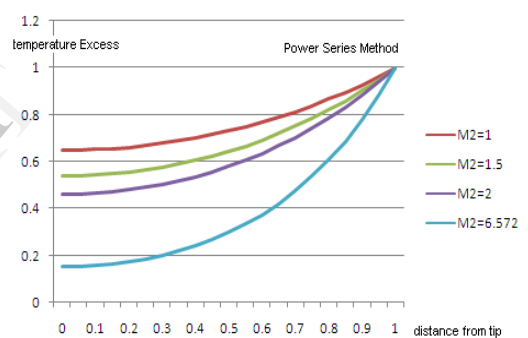


Figure-4: temperature distribution at various  $M^2$ .

**Differential Transformation Method (DTM)**

**J.K Zhou** proposed concept of DTM. This method Construct an analytical solution in the form of data functions. Differential transform method can easily be applied to DAEs and series solutions are obtained. After the transformation, here, we have formulated series coefficients very simply for the considered problems. Steady state heat conduction in fin profile using a exact series method of solution called differential transformation method. It converges with only six or less terms .it does not require use of Bessel or other special functions.

Original function y(x)	Transformed function Y(k)
u(x)±v(x)	U(k)±V(K)
cw(x)	cW(k)
$\frac{dw}{dx}$	(k+1)W(k+1)
$\frac{d^2w}{dx^2}$	(k+1)(k+2)...(k+n)W(k+n)
$\frac{d^n w}{dx^n}$	$\sum_{l=0}^k U(l)V(k-l)$
u(x)v(x)	$\delta(k-n) = \begin{cases} 1, k = n \\ 0, k \neq n \end{cases}$
$x^n$	

Table-3- Fundamental operations of 1D Differential Transformation Method.

### 7. CONCLUSIONS

The differential Transformation technique is one of the numerical methods for ordinary deferential equations, which uses the form of polynomials as the approximation to the exact solutions that are sufficiently differentiable and it provides iterative procedure to obtain the high-order Taylor series. Taylor series method is computationally taken long time for large orders. The differential transform is an iterative procedure for obtaining Taylor series solutions of differential equations and system of PDEs. This method reduces the size of computational domain and applicable to many problems easily. Power series and FDM are powerful methods. But the modern method such as Differential Transformation Method (DTM) is very fast to calculate and gives reliable and accurate result.

These transformations are then applied to the differential equation given above. Differential transform of

$$\theta(x) = \frac{1}{k!} \left( \frac{d^k \theta}{dx^k} \right)_{x=0} = T(k)$$

Then inverse transformation  $\theta(x) = \sum_{k=0}^{\infty} x^k T(k)$

$\theta(x) = T(0) + xT(1) + x^2T(2) + x^3T(3) + x^4T(4) + \dots$  Now transform each term of equation For various values of T (k), the dimensionless temperature distribution is given by

$$\theta(x) = T(0) + xT(1) + x^2T(2) + x^3T(3) + x^4T(4) + \dots$$

$$\theta(x) = 0.64863 + 0.32431x^2 + 0.0270x^4 + \dots$$

(1)	(2)	(3)	(4)	(5)
X	M <sup>2</sup> =1	M <sup>2</sup> =1.5	M <sup>2</sup> =2	M <sup>2</sup> =6.572
0	0.648064	0.540997	0.459184	0.154324
0.2	0.661069	0.557309	0.477674	0.175057
0.4	0.700604	0.607226	0.534633	0.24282
0.6	0.768258	0.693758	0.634646	0.375715
0.8	0.866741	0.822115	0.785746	0.608579
1	1	1	1	1

Table 4 - Temperature distribution for Rectangular fin by DTM.

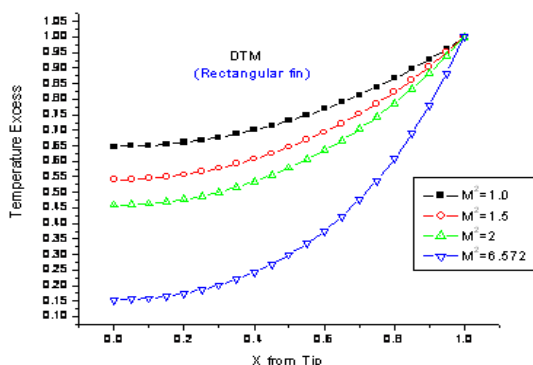


Figure 5: Temperature excess by DTM at various M<sup>2</sup>

### ACKNOWLEDGEMENT

Most responsible for the direction of the research conducted during this work is Dr. KVS Rao, professor, Department of Mechanical, University College of engineering, RTU Kota and Dr. alka Agarwal, professor & head University Institute of technology RGPV. I will always be grateful for their efforts and also thankful for Crucial guidance and facilities were provided them.

Finally, I thank to my family members and colleagues who supported and encourage me to complete this work.

### REFERENCES

1. Abdel-Halim Hassan IH., "Differential transformation technique for solving higher-order initial value problems". Appl Math Comput 2004; 154:299-311.
2. B.A. Brigham, G.J. Vanfossen, "Length to diameter ratio and row number effects in short pin fin heat transfer", ASME J. Eng. Gas Turbines Power 106 (1984) 241-244.
3. B. Kundu, "Analysis of thermal performance and optimization of concentric circular fins under dehumidifying conditions", International Journal of Heat and Mass Transfer 52 (2009) 2646-2659.
4. B. Kundu\*, D. Bhanja, "Performance and optimization analysis of a constructal T-shaped fin subject to variable thermal conductivity and convective heat transfer coefficient", International Journal of Heat and Mass Transfer 53 (2010) 254-267.
5. Balaram Kundu, Dipankar Bhanja, Kwan-Soo Lee, "A model on the basis of analytics for computing maximum heat transfer in porous fins", International Journal of Heat and Mass Transfer (2012).
6. Ghai M.L., "Heat transfer in straight fins", Proc. Gen. Disc. Heat Transfer, Institute of Mechanical Engineers, London, 1951, p.180.
7. G.J. Vanfossen, "Heat-transfer coefficients for staggered arrays of short pin fins", ASME J. Eng. Power 104 (1982) 268-274.
8. Holman J.P., Heat Transfer, 9th ed., McGraw-Hill, 2002.
9. H.C. U"nal, "Determination of the temperature distribution in an extended surface with a non-uniform heat transfer

- coefficient*", Int. J. Heat Mass Transfer 28 (12) (1985) 2279–2284.
10. H.C. U'nal, "An analytic study of boiling heat transfer from a fin", Int. J. Heat Mass Transfer 30 (2) (1987) 341–349.
  11. I.N. Dul'kin \*, G.I. Garas'ko, "Analysis of the 1-D heat conduction problem for a single fin with temperature dependent heat transfer coefficient: Part I – Extended inverse and direct solutions", International Journal of Heat and Mass Transfer 51 (2008) 3309–3324.
  12. Incropera F.P., DeWitt D.P., "Fundamentals of Heat and Mass Transfer", 5th ed., John Wiley & Sons, 2002.
  13. Kern D.Q., Kraus D.A., "Extended Surface Heat Transfer", McGraw-Hill, New York, 1972.
  14. Tzer-Ming Jeng \*, Sheng-Chung Tzeng, "Pressure drop and heat transfer of square pin-fin arrays in in-line and staggered arrangements", International Journal of Heat and Mass Transfer 50 (2007) 2364–2375.

IJERT