

Fabrication of Automatic Speed Bump with Day Night Control

S Mathivanan¹ Z Mohamed yaseen² M Murugavel²

P Saravanan² S Vinothkumar

²¹-Assistant Professor,

²-UG Scholar

Department of Mechanical Engineering, Hindusthan Institute of Technology, Coimbatore-32.

Abstract : Here we are fabricating the model pneumatic speed breaker with day night control. Nowadays traffic has increased as the use of vehicle has increasing day to day. We require speed breaker to slow down the vehicles which are traveling at high speed. In certain areas at daytime traffic may be heavier than nighttime. So, we require speed brake only during the daytime and not at night. For this purpose, this project gives a solution. This equipment consists of pneumatic cylinder, Speed breaker setup, solenoid valve, LDR, proximity Sensor and Control unit.



Fig : Automatic Speed Bump

1. INTRODUCTION

According to the NATPAK report of road accidents 2009, India ranks 2nd in the number of persons killed in road accidents and among the Indian states Kerala ranks 2nd. According to their findings, carelessness of the drivers, improper and nonscientific construction of roads and speed breakers, over speeding and ineffective traffic management contributes about 72% of the total number of accidents.

Our project automatic speed breaker is a new concept in this field and its innovative too. The device mainly consists of a speed breaker which is operated with the help of electric power. This system is mainly employed in the areas where the need of speed breaker is restricted to certain specific timings in areas like school and collage roads, theatre roads etc. and during the other hours the inconvenience of the speed breaker can be removed by folding down the speed breaker below the road surface. Hence they seem to be more effective in against over speeding and helps in traffic management.

2. LITERATURE REVIEW

There is evidence to support the claim that speed breakers can cause accidents and injury when a vehicle approaches a speed-breaker at a speed greater than some threshold velocity, the risk accident or injury is substantial. Speed- breakers are inconspicuous in low

visibility conditions, like at night, or when there is fog, rain or snow. This problem is particularly acute in developing countries where speed-breakers don't always accompany warning signs. We propose an early warning system that uses a Smartphone based application to alert the driver in advance when the vehicle is approaching a speed breaker. In addition, the application constantly monitors the Smartphone accelerometer to detect previously unknown speed-breakers. The proposed detection algorithm easy

implement because it does not require accelerometer reorientation this is one of the main contributions of our work since previous approaches have used expensive computations to reorient the accelerometer. The algorithm was evaluated using 678 Km of drive data, which involved 22 different drivers, 5 different types of vehicles (bus, auto rickshaw, cycle rickshaw, motorcycle, and car), and 4 smart phones. The results are very promising and can be further improved by aggregating detection reports from multiple smart phones.

3. DESIGN CALCULATION

DESIGN OF PISTON ROD:

Load due to
air Pressure.

P

Diameter of = 40 mm

the Piston

(d)

Pressure = 6 kgf/cm²

acting (p)

Material = C-45

used for rod

factor of

safety

Force acting = Pressure x Area
on the rod

(P)

$$= p \times (\pi d^2 / 4)$$

$$= 6 \times \{(\pi \times 4^2) / 4\}$$

$$P = 73.36 \text{ Kgf}$$

$$= \sigma_y / \text{FOS}$$

Design

Stress (σ_y)

$$= 36 / 2$$

$$= 18 \text{ Kgf/mm}^2$$

$$= P / (\pi d^2 / 4)$$

$$\therefore d = \sqrt{4 p / \pi [\sigma_y]}$$

$$= \sqrt{4 \times 75.36 / \{\pi \times 18\}}$$

$$= \sqrt{5.33}$$

$$= 2.3 \text{ mm}$$

\therefore Minimum diameter of rod required for the load = 2.3 mm

DESIGN OF CYLINDER

THICKNESS:

Material used = Cast iron

Assuming = 40 mm

internal

diameter of the cylinder

Ultimate tensile stress = 250 N/mm²

Working Stress = 2500gf/mm²
= Ultimate tensile stress / factor of safety

Assuming factor of safety = 4

Working stress (ft) = 2500 / 4
= 625 Kgf/cm²

According to 'LAMES EQUATION'

Minimum thickness of cylinder (t) = $r_i \{ \sqrt{(f_t + p)/(f_t - p)} - 1 \}$

Where,

r_i = inner radius of cylinder in cm.

f_t = Working stress (Kgf/cm²)

p = Working pressure in Kgf/cm²

∴
Substituting values we get,

$$t = 2.0 \{ \sqrt{(625+6)/(625-6)} - 1 \}$$

$$t = 0.019 \text{ cm} = 0.19 \text{ mm}$$

We assume thickness of cylinder = 2.5 mm

Inner diameter of barrel

$$= 40 \text{ mm}$$

Outer diameter of barrel

$$= 40 + 2t$$

diameter of barrel

$$= 40$$

$$+ (2 \times 2.5) = 45 \text{ mm}$$

DESIGN OF PISTON ROD

DAIMETER OF PISTON ROD:

$$\text{Force of piston} = \text{Pressure} \times \text{area} = p \times \Pi/4 (d^2)$$

$$= p \times \Pi/4 (d^2)$$

n

Rod

(P)

$$= 6 \times (\Pi/4) \times (4)^2$$

$$= 73.36 \text{ Kgf}$$

$$\text{Also, force} = (\Pi/4) (d_p)^2 \times f_t$$

on piston rod

(P)

$$P = (\Pi/4) \times (d_p)^2 \times 625$$

$$73.36 = (\Pi/4) \times (d_p)^2 \times 625$$

$$\therefore dp^2 = 73.36 \times (4/\pi) \times (1/625) = 0.15$$

$$dp = 0.38 \text{ cm} = 3.8 \text{ mm}$$

$$\text{By standardizing } dp = 15 \text{ mm}$$

Stroke length : Cylinder stoker length
160 mm = 0.16 m
Quantity : 1

Seals : Nitride (Buna-N) Elastomer

LENGTH OF THE PISTON ROD:

Approach stroke = 160 mm
Length of threads = $2 \times 20 = 40 \text{ mm}$
Extra length due to front cover = 12 mm
Extra length of accommodate head = 20 mm

Total length of the piston rod = $160 + 40 + 12 + 20 = 232 \text{ mm}$

By standardizing, length of the piston rod = 230 mm

End cones : Cast iron

Piston : EN – 8
Media : Air
Temperature : 0-80 ° C
Pressure : 8 N/m²
Range

SINGLE ACTING PNEUMATIC CYLINDER:

Stroke length : Cylinder stoker length
80 mm = 0.08 m

Quantity : 2
Seals : Nitride (Buna-N) Elastomer

End cones : Cast iron

Piston : EN – 8
Media : Air
Temperature : 0-80 ° C

SPECIFICATION

DOUBLE ACTING PNEUMATIC CYLINDER:

Pressure : 8 N/m²
Range

SOLENOID VALVE

Max pressure : 0-10 x 10⁵
range N/m²

FLOW CONTROL VALVE

Port size : 0.635 x
10⁻² m

Pressure : 0-8 x 10⁵
N/m²

Media : Air

Quantity : 1

CONNECTORS

Max working : 10 x 10⁵
pressure N/m²

Temperature : 0-100 °
C

Fluid media : Air

Material : Brass

HOSES

Max pressure : 10 x 10⁵
N/m²

Outer diameter : 6 mm
= 6 x 10⁻³ m

Inner diameter : 3.5 mm
= 3.5 x 10⁻³ m

4.14 FORCE CALCULATION OF PNEUMATIC CYLINDER

Force to be exerted is 40N

Force = pressure x area

Pressure = 0.4 x 10⁵ N/m²

in the
cylinder

Area of = Force/pressure
the piston,

(πd^2)/4

= 40/ 40000

= 0.001m²

Bore = 0.0356m = 35.6
diameter mm

FOR FORWARD STROKE

For 40mm

bore diameter

Corresponding = 16mm

rod diameter

Area of the = (πd^2)/4

piston

= ($\pi \times 40^2$)/4

=

1256.8mm²

Force = pressure x
(modified) to area
be exerted

$$\begin{aligned}
 &= 0.4 \times 10^5 \\
 &\times 1256.8 \\
 &= 50\text{N}
 \end{aligned}$$

$$\begin{aligned}
 \text{Extending force} &= 50.3 \text{ N} \\
 \text{Retracting force} &= 42.2 \text{ N}
 \end{aligned}$$

FOR RETURN STROKE

On the return stroke, when the pressure is applied to the reverse direction, the force on the piston due to the pressure is $= P \times (A-a)$

Where,

P = Pressure in the cylinder (N/m²)

A = Area of the piston (m²)

a = Cross sectional area of the piston rod (m²)

Therefore,

$$\begin{aligned}
 \text{Area of the piston} &= \{(\pi \times d^2)/4\} - \{(\pi \times d_1^2)/4\} \\
 (A-a) &= \{(\pi \times 40^2)/4\} - \{(\pi \times 16^2)/4\} \\
 &= 1256.6 - 201 \\
 &= 1055 \text{ mm}^2
 \end{aligned}$$

FORCE TO BE CONVERTED

$$\begin{aligned}
 \text{On the reverse direction} &= \text{pressure} \times \text{area}
 \end{aligned}$$

$$\begin{aligned}
 &= 0.4 \times 10^5 \times 1055 \\
 &= 42.2 \text{ N}
 \end{aligned}$$

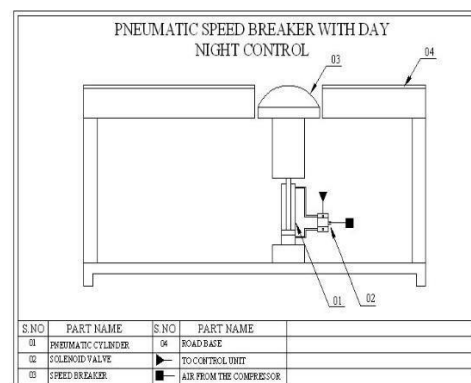
For working pressure of $0.4 \times 10^5 \text{ N/m}^2$

4. WORKING PRINCIPLE

The speed breaker works normally during the day time on to the roads this speed breaker setup consists of the pneumatic cylinder which has a piston for the linear motion of the speed breaker. This piston is connected to the speed breaker and when the piston moves in the downward direction, the speed breaker moves down so that the road is made free for the speeding.

This project has a sensor and the control unit which operates on the light rays, during the day time the sunrays falls on the LDR sensor unit, the sensor sends the signal to the control unit and the control unit actuates the pneumatic cylinder and then the cylinder piston moves forward direction and makes the speed breaker on the road. During the night time, the LDR sensor does not get any light signal from the sun and it deactivates the control unit which makes the piston of the pneumatic cylinder to retract making the speed breaker to get down and thus making the road speed breaker free on the night time.

5. LAYOUT OF THE MODEL



6. CONCLUSION

A strong multidiscipline team with a good engineering base is necessary for the Development and refinement of advanced computer programming, editing techniques, diagnostic Software, algorithms for the dynamic exchange of information at different levels of hierarchy.

This project work has provided us an excellent opportunity and experience, to use our limited knowledge. We gained a lot of practical knowledge regarding, planning, purchasing, assembling, and machining while doing this project work.

We are proud that we have completed the work with the limited time successfully. The **“FABRICATION OF AUTOMATIC SPEED BUMP WITH DAY NIGHT CONTROL”** is working with satisfactory conditions. We are able to understand the difficulties in maintaining the tolerances and also quality.

We have done to our ability and skill making maximum use of available facilities. In conclusion remarks of our project work. Thus, we have developed a **“AUTOMATIC SPEED BUMP WITH DAY NIGHT CONTROL”**. By using more techniques, they can be modified and developed according to the applications.

7. REFERENCES

- [1] **AFUKAAR, F.** Speed control in developing countries: Issues, Challenges and opportunities in reducing road traffic injuries. Injury Control and Safety Promotion 10, 2 (2003), 77–81.
- [2] **ASLAN, S., KARCIOGLU, O., KATIRCI, Y., KANDI, H., EZIRMIK, N., AND BILIR, O.** Speed bump induced spinal column injury. The American Journal of Emergency Medicine 23, 4 (2005), 563 – 564.
- [3] **ZAIDEL, D., HAKKERT, A., AND PISTINER, A.** The use of road
Humps for moderating speeds on urban streets.
Accident Analysis
And Prevention 24, 1 (1992), 45 – 56
- [4] **MUNJIN, M. A., ZAMORANO, J.J., MARRE, B., ILABACA, F., BALLESTEROS, V., MARTINEZ, C., YURAC, R., URZUA, A., LECAROS, M., FLEIDERMAN, J., AND GARCIA, N.** Speed Hump spine fractures: Injury mechanism and case series. Journal Of Spinal Disorders & Techniques 24, 6 (2011)
- [5] <https://www.researchgate.net/topic/Engineering>
- [6] <https://www.irjet.net/>