

A Comprehensive Study on Integration Technique on Air Separation Unit and Gas Turbine for IGCC Power Plant

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Abstract— In recent decades, the Chinese economy has been increasing after rapid industrial development, urbanization, and education. Alongside the amelioration, China has to face the stringent environmental policy being a significant emitter of SO₂, CO₂, and particulate matter. To overcome the effects of all such pollutants, Integrated Gasification Combined Cycle (IGCC) Technology had proved to be a boon to meet the environmental regulations. IGCC ensures the green use of coal by heating coal in a gasifier with oxygen and steam, producing syngas free from sulfur and carbon dioxide; thus, sustainable energy attained through Bryton cycle and Rankin cycle. For the proposed model, a process simulation tool, EBSILON Professional is considered. The study focused on the integration of gas turbine (GT) with an air separation unit (ASU) at high pressure as it plays a vital role to increase the net efficiency of the plant. For ASU-GT integration, two case studies are considered; 1) the injection rate of N₂ with air extraction rate from GT compressor at full, partial and without load, 2) the extraction rate of air with N₂ injection at full, partial and without were assayed. The results suggest that the operation of separate ASU or without N₂ infusion were not advantageous. However, the N₂ injection increased the power output for both the GT and the (ST). Among the solutions of various integration conditions, the most suitable combinations for the cases mentioned in above suggested by comparing them with the base case; where air extraction rate at 80% and 100% with nitrogen induction at 100% (C17, C18) respectively and nitrogen induction rate at 80% and 100% with 95% air extraction (C34, C35) The mass flow rate and turbine inlet temperature (TIT) plays a vital role in the performance of GT. Moreover, no separate compressor designed for further compression of N₂ to GT combustor.

Keywords: Air separation unit, air extraction, gas turbine, IGCC

I. INTRODUCTION

The economic growth of China over the last three decades has been remarkable; its dilation has reached its acme and better amelioration in the use of technology. Over this period, the total energy consumption increased up to 70% with 75% increment in coal utilization [1]. Due to which environmental degradation has become the foremost problem not only for China but also for the entire world as it causes global warming, health issues, and the economy. On average,

environmental degradation is causing more demise of people than AIDS, breast cancer, tuberculosis, and malaria [2]. Therefore, such inevitable facts ensure the diminution of coal burning for electricity generation. Now, China is the world's largest developing country with extreme consumption of coal and is a carbon emitter. Without diminution, the CO₂ production may reach its summit to about 50% for the following 15 years [3]. At present, 40% of the world's energy is fueled by the coal burning and is forecasted to supply a strategic share for over the next three decades [4]. Due to the availability of coal for eons, the most expedite technology for the clean burning of coal is IGCC with carbon capture and storage (CCS) which may increase its efficiency to more than 39%. The IGCC technology is far better than pulverized coal combustion (PCC), fluidized bed combustion (FBC) and ultra-supercritical technology due to low emissions of NO_x, SO_x, 20% CO₂ production and 20-40% less use of water [5]. Also, IGCC technology is suitable for a wide variety of coal, whether dry feed or wet feed. The IGCC technology contains various process sections like gasification, syngas cooling, sulfur recovery, CO₂ sequestration, GT, heat recovery, and steam generation (HRSG). Ongoing research on IGCC focused on thermal efficiency and design considerations. Keeping in consideration all the benefits from this technology, China owns only two IGCC plants, and none of them proved to be beneficial.

II. LITERATURE REVIEW

According to NETL 2007, coal is the primary raw material for IGCC power plants followed by pet coke, biomass, and petroleum as a by-product. Moreover, co-gasification is the most suitable option for electricity generation with such type of power plants. In general, IGCC technology is inherited with complex integration problems and dependency on the various unit process within the plant; thus, providing a big gap for the improvement. The studies on IGCC technology is categorized into three major sections 1) exergy analysis, 2) economic analysis, 3) performance analysis. In all three analysis, IGCC technology is examined with CCS, IGCC with a solid oxidized fuel cell (SOFC) and IGCC with co-gasification. The studies on the optimization by taking into

the exergy analysis can be found in the literature [6-10], where various techniques are considered like extended exergy accounting methodology and exergy analysis with Selexol processes (MEDA, MEA, Methanol, etc.). The studies on the optimization by choosing various modeling approach are [11-15]. Similarly, lots of research work is done on IGCC with CCS as indicated in literature [9,16-20], and the hybridization of IGCC with other renewable technologies can be found in various studies [21-23]. A major portion of research focused on the co-gasification for the production of syngas for different chemicals manufacturing, hydrogen production and electricity production with IGCC [24-30]. The integration of cryogenic ASU with GT has been identified as a key process because ASU normally consumes a large amount of power [31] and this integration has a significant impact on total power consumption as well as system net efficiency [32]. The integration of ASU leads to a smooth flow of air and syngas to the GT while reducing NO_x formation. There are three types of technologies available for the generation of high purity oxygen that includes pressure swing adsorption, cryogenic distillation, and membrane technology. Above all, the cryogenic technology is the best one for the IGCC where a large amount of oxygen is required to gasify the coal. The power consumption of the ASU unit is very high ultimately affecting the overall efficiency of IGCC [33]. Table 1 illustrates the compendium studies on ASU-GT integration.

III. PROBLEM FORMULATION

This study intends to evaluate the ASU-GT unit and find the optimal solution for the most efficient, environmentally friendly, and economical key for commercial IGCC technology with CCS. There are three types of technology available for the production of high purity oxygen by pressure swing adsorption, cryogenic distillation, and membrane technique [32]. Above all, the cryogenic technology is the best one for the IGCC, where a significant amount of oxygen is required to gasify the coal [33]. Therefore, some of the studies conducted on ASU-GT integration as illustrated in Table 1 need further improvements for this unit. Moreover, the elevated pressure cryogenic EP-ASU was used for the production of O₂ and N₂ as it operates at the pressure equal to the pressure of the GT compressor [34]. The EP-ASU technologies are most preferred when air extraction from the GT compressor is considered. As the ASU is operating at high pressure, it favors the smooth suction for O₂ production to their final delivery pressure; thus, reducing power consumption.

Table 2 shows that the 9H-GE GT is selected for the firing of syngas and is the most suitable GT with high efficiency and low environmental indicators [38]. The proposed model is simulated on EBSILON Professional; the most suitable and appropriate software for all types of power plants including IGCC, SOFC, and solar power plant with all kinds of Rankin cycle as listed in literature [39-42].

Table 1: A Literature review on ASU-GT unit for IGCC power plant

| Authors | Major results |
|-------------------------|--|
| Wimer et al. 2006 [34] | The study conducted on air extraction and N ₂ injection by cryogenic ASU technology with low pressure and elevated pressure and then comparing the results with membrane-based ASU technology. |
| Emun et al. 2009 [5] | The IGCC power plant simulated on Aspen Plus with an increment in efficiency by operating ASU at 19 bar with an N ₂ injection to GT at 55% only. However, the study was not conducted on air extraction rate. |
| Cormos et al. 2010 [35] | Hydrogen and electricity production by IGCC 400 MW and CCS 90% carbon capture rate using Aspen Plus. |
| | The thermal efficiency and energy consumption compared to simulating different cases including the sensitivity analysis |
| | The extraction of air from the GT compressor is studied only for Case-1 with ASU unit operating at 0.237 MPa. |
| He et al. 2012 [36] | A mathematical model is developed to study the part load performance of syngas burning in GT and comparing it to the base case, i.e., the natural gas turbine. |
| | The impacts of the variable inlet guide, degree of air extraction, and the degree of fuel dilution were studied. |
| Lee et al. 2012 [37] | The study contains mathematical optimization called self-optimized control (SOC) for the elevated pressure (EP ASU) using gPROMS simulation software. |
| | The study includes the integration of ASU-GT unit by considering air extraction rate (27.7% & 28%), steam injection rate (17.3% & 18.2%) and N ₂ injection rate (71% & 69.2%) only. |
| | The study does not include the comprehensive integration of ASU-GT unit, and for N ₂ injection to GT, a separate compressor utilized as ASU was operating a low pressure. |
| Han et al. 2017 [32] | The integration of ASU-GT unit simulated on Thermoflex software where the detailed study conducted on air extraction rate along with N ₂ induction rate. |
| | Moreover, the ASU unit operated at high pressure, and for N ₂ injection to GT, a separate compressor is utilized. |

Table 2: Design specification of 9H-GE gas turbine [38]

| Output (MW) | Inlet temperature (°C) | Air flow (kg/sec) | Pressure ratio |
|-------------|------------------------|-------------------|----------------|
| 480 | 1430 | 685 | 23 |

The study includes the air extraction rate from the GT and simultaneous operation of ASU. So, this proposed model considered two base cases: 1) the extraction of air from GT compressor (with N₂ injection to GT) by running the ASU at without, partial and full load, 2) the injection rate of N₂ to GTR by extracting air form GT compressor at without, partial and full load. The extracted air is being utilized for O₂ production from ASU to minimize energy consumption. Fig 1 explains the integration technique for ASU-GT section; where the air is removed from the GT compressor, and N₂ is injected to GT combustor for different integration combinations. The obtained results are compared with the base case and selected for the most optimized operation of the IGCC power plant. In addition to air extraction, the N₂ injection causes a remarkable increment to the overall performance of the IGCC plant by controlling the adiabatic temperature of the flame and reduce the NO_x formation in GT [5,32].

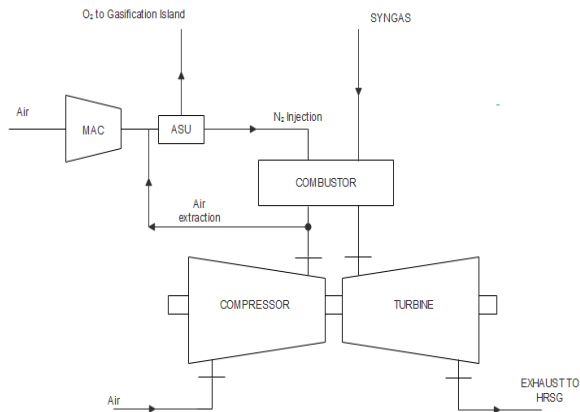


Fig 1: The schematic diagram of ASU-GT unit

Figure 2 explains the various components of IGCC power plant simulated on EBSILON Professional for syngas production, syngas cooling with steam production, CO₂ sequestration, GT combustor, Bryton cycle, Rankin cycle, and HRSG units. Although, the application of gasifier is universal for all types of feedstock however dry feed is more beneficial over slurry feed as it lists to agglomeration or clogging of coal thus demands more oxygen and energy consumption [43,44]. The proposed model of IGCC is simulated with EBSILON Professional to generate more efficient and environmentally friendly model [45]. The software is equipped with all types of unit operations like heat exchangers, compressors, pumps, steam turbines, gas turbines, and generators. The EBSILON performs the calculation of the whole process by taking into consideration of energy and mass balance laws for each species, chemical equilibrium, and pressures laws. EBSILON Professional is specifically designed for processing and monitoring of available data, economic evaluation to deviations by providing spot errors, malfunction, and areas of improvement. The simulation for a complex system like power plant is beneficial especially when it comes to studying IGCC technology which involves many chemical and mechanical technologies providing total performance calculations, detailed analysis on chemical quantities and thermodynamic calculations, studying variations in typology and parameters for the whole process.

The assumptions for the simulation model are as follow:

- The model is at steady-state, and the kinetics for char gasification and combustion are included.
- The model considers all the processes occurring in the gasifier, i.e., the coal drying, the coal pyrolysis, the char gasification, and the char combustion. Coal drying and pyrolysis take place instantaneously at the top of the gasifier.
- The solid and gas phase flow in a plug-flow pattern and the pressure drop in the gasifier is neglected.
- The variable bed void-age throughout the gasifier is taken into consideration and temperature of the solid, and the gas is equal inside the gasifier.

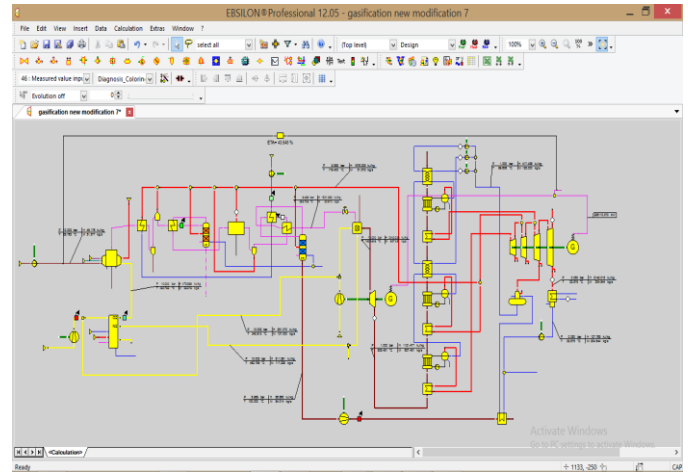


Figure 2: The Process flow diagram of the IGCC power plant by EBSILON Professional

At the initial stage, coal is introduced to the gasification unit as a dry feed with the help of high-pressure nitrogen at 48 bar. The ASU is primarily used to supply oxygen for gasification reaction, and nitrogen was also provided to the combustion chamber of GT. The inlet temperature of steam and oxygen was 600⁰C and 189⁰C, respectively [43,46]. The syngas produced at equilibrium temperature 1531⁰C and 48 bar pressure [47,48] after rejecting heat and building steam. The cooled syngas enters the H₂S removal plant where sulfur production takes place by clause process.

The high-pressure syngas enters the water gas shift reactor for CO conversion, and finally, CO₂ sequestration takes place. The purified syngas enters the GT combustor at 300⁰C for smooth burning and electricity production. After the gasification of coal, the high-temperature syngas enters the gas cooling section where high-temperature steam produced for power generation in the Rankine cycle. The cooled gas comes H₂S removal plant and CO₂ sequestration plant wherein both cases 95% separation of said elements takes place. The water gas shift reactor used at high temperature for the conversion of CO to CO₂. The next station for syngas is CO₂ sequestration process where the intense amount of gas separates depending upon the composition of coal. The syngas rich in high content of H₂ and CO enters the combustion chamber to generate high-temperature flue gas. The HRSG and combustion chamber produce steam and power from the syngas thermodynamic reaction and hot flue gas from GT combustor. The HRSG unit provides three stages of steam, i.e., high pressure, intermediate pressure, and low pressure for the steam cycle.

IV. METHODOLOGY

The study on ASU-GT conducted by considering six integration indicators which are explained by the following equations. The air extraction rate is defined to be the ratio of extracted air from GT to the total air flow entering ASU given by the following equation;

$$X_{Air} = \frac{Air_{ex}}{Air_{ex} + A_{ASU}} \times 100 \quad (1)$$

Where;

Air_{ex} = rate of air extraction from the GT compressor

Air_{ASU} = Inlet of air from the separate compressor

E_{GI} = the energy produced by the gasification island

E_{PI} = the electricity generated by the power generation island

$$E_{net} = E_{GT} + E_{ST} - E_{ASU} - E_{aux} \quad (4)$$

Table 3 Study on air extraction rate at without, partial and full injection of N₂

| Description | BC | C-1 | C-2 | C-3 | C-4 | C-5 | C-6 |
|--|-------|--------|--------|--------|--------|--------|--------|
| X_{air} | 0 | 20 | 40 | 50 | 60 | 80 | 100 |
| X_{N_2} | 0 | 0 | 0 | 0 | 0 | 0 | 0 |
| Extracted Air kg/sec | 0 | 33.56 | 66.44 | 83.63 | 100.4 | 133.5 | 160.7 |
| Air from compressor kg/sec | 685 | 685 | 685 | 684 | 685 | 685 | 685 |
| Syngas flow kg/sec | 30.09 | 29.38 | 28.71 | 28.37 | 28.01 | 27.31 | 26.75 |
| Exhaust gas inlet temperature GT °C | 1430 | 1429 | 1429 | 1430 | 1430 | 1429 | 1429 |
| Coal thermal energy LHV MW _{th} | 1258 | 1216 | 1177 | 1151 | 1136 | 1095 | 1063 |
| GT output (MW) | 341.6 | 313.1 | 286.4 | 272.2 | 258.3 | 231.6 | 208.7 |
| ST output (MW) | 213.3 | 216.4 | 219.8 | 220.9 | 222.2 | 225.5 | 227.2 |
| Gross output (MW) | 554 | 529 | 506.2 | 493.1 | 480.5 | 457.1 | 435.9 |
| Total Auxiliary (MW) | 153.5 | 129.2 | 110.3 | 100.5 | 89.5 | 62.1 | 31.5 |
| Net output | 401.8 | 400.8 | 395.7 | 393.2 | 391.7 | 394.1 | 405.0 |
| Gross efficiency % | 44.11 | 43.55 | 42.97 | 42.65 | 42.34 | 41.64 | 41.06 |
| Net efficiency % | 31.92 | 32.93 | 33.60 | 33.97 | 34.46 | 35.97 | 38.09 |
| Description | BC | C-7 | C-8 | C-9 | C-10 | C-11 | C-12 |
| X_{air} | 0 | 0 | 20 | 40 | 60 | 80 | 100 |
| X_{N_2} | 0 | 50 | 50 | 50 | 50 | 50 | 50 |
| Extracted Air kg/sec | 0 | 33.56 | 66.44 | 83.63 | 100.4 | 133.5 | 160.7 |
| Air from the compressor kg/sec | 685 | 685 | 685 | 685 | 685 | 685 | 685 |
| Syngas flow kg/sec | 30.09 | 30.91 | 30.30 | 29.66 | 29.01 | 28.39 | 27.95 |
| Exhaust gas inlet temperature GT °C | 1430 | 1430 | 1430 | 1430 | 1430 | 1429 | 1429 |
| Coal thermal energy LHV MW _{th} | 1258 | 1216 | 1177 | 1157 | 1136 | 1095 | 1063 |
| GT output | 341.6 | 390.2 | 362.8 | 335.6 | 307.8 | 280.0 | 258.3 |
| ST output | 213.3 | 209.7 | 212.6 | 215.5 | 218.2 | 220.7 | 222.5 |
| Gross output | 554.9 | 604.9 | 575.4 | 551.1 | 526 | 500.7 | 480.8 |
| Total Auxiliary MW | 153.5 | 154.6 | 130.4 | 111.6 | 90.84 | 63.30 | 33.07 |
| Net output | 401.8 | 445.5 | 445.0 | 439.5 | 435.3 | 437.5 | 447.8 |
| Gross efficiency % | 44.11 | 47.18 | 45.41 | 44.69 | 44.02 | 43.22 | 42.43 |
| Net efficiency % | 31.92 | 35.02 | 35.00 | 35.64 | 36.42 | 37.76 | 39.51 |
| Description | BC | C-13 | C-14 | C-15 | C-16 | C-17 | C-18 |
| X_{air} | 0 | 0 | 20 | 40 | 60 | 80 | 100 |
| X_{N_2} | 0 | 100 | 100 | 100 | 100 | 100 | 100 |
| Extracted Air kg/sec | 0 | 33.56 | 66.44 | 83.63 | 100.4 | 133.5 | 160.7 |
| Air from the compressor kg/sec | 685 | 685 | 685 | 685 | 685 | 685 | 685 |
| Syngas flow kg/sec | 30.09 | 31.70 | 31.17 | 30.58 | 30.01 | 29.49 | 29.15 |
| Exhaust gas inlet temperature GT °C | 1430 | 1430.1 | 1430.2 | 1429.8 | 1431.6 | 1431.2 | 1429.5 |
| Coal thermal energy LHV MW _{th} | 1258 | 1355 | 1323 | 1287 | 1253 | 1223 | 1203 |
| GT output (MW) | 341.6 | 438.2 | 411.0 | 384.0 | 357.0 | 329.9 | 307.8 |
| ST output (MW) | 213.3 | 205.9 | 208.5 | 211.2 | 214.0 | 216.2 | 217.4 |
| Gross output (MW) | 554.9 | 644.1 | 649.5 | 595.2 | 571 | 546.1 | 525.2 |
| Total Auxiliary (MW) | 153.5 | 155.6 | 131.6 | 112.8 | 92.14 | 64.73 | 34.62 |
| Net output (MW) | 401.8 | 488.5 | 487.9 | 482.5 | 478.9 | 481.4 | 490.6 |
| Gross efficiency % | 44.11 | 47.52 | 46.82 | 46.23 | 45.54 | 44.64 | 43.65 |
| Net efficiency % | 31.92 | 36.04 | 36.88 | 37.47 | 38.19 | 39.35 | 40.77 |

$$IN_{nitrogen} = \frac{N_{2,in}}{N_{2,total}} \times 100 \quad (2)$$

Where;

$N_{2,in}$ = Injection of air to the GT combustor from the ASU

$N_{2,total}$ = the total amount of N₂ produced by ASU

$$E_{aux} = E_{GI} + E_{CO_2} + E_{PI} \quad (3)$$

Where;

E_{aux} = the amount of power utilized by all auxiliaries equipment is within in IGCC plant except ASU

Where;

E_{net} = the net electricity provided by the proposed system

E_{GT} = the electricity generated by the GT

E_{ST} = the electricity generated by the ST

E_{ASU} = the power consumed by the ASU

E_{aux} = the power consumed by the remaining auxiliaries

$$\eta_{net} = \frac{E_{net}}{E_{Coal}} \times 100$$

The net efficiency is defined as the ratio of energy produced by the plant to the thermal energy of the coal.

Where E_{net} is the net electricity generated by the plant and

E_{coal} is the thermal energy of Coal.

Table 4 Study on nitrogen injection rate at full, partial of air extraction

| Description | BC | C-19 | C-20 | C-21 | C-22 | C-23 | C-24 |
|---|--------|--------|--------|--------|--------|--------|--------|
| X_{air} | 0 | 0 | 0 | 0 | 0 | 0 | 0 |
| X_{N_2} | 0 | 20 | 40 | 50 | 60 | 80 | 100 |
| Extracted air kg/sec | 0 | 0 | 0 | 0 | 0 | 0 | 0 |
| Air from compressor kg/sec | 685 | 685 | 685 | 685 | 685 | 685 | 685 |
| Syngas flow kg/sec | 30.09 | 30.44 | 30.76 | 30.91 | 31.08 | 31.38 | 31.69 |
| Exhaust gas inlet temperature GT °C | 1430.3 | 1431.5 | 1431.4 | 1430.7 | 1431.2 | 1429.3 | 1429.5 |
| Coal thermal energy LHV (MW _{th}) | 1258.2 | 1279.3 | 1298.6 | 1307.6 | 1317.9 | 1335.7 | 1354.9 |
| GT output (MW) | 341 | 361.5 | 380.88 | 390.23 | 400.1 | 418.7 | 437.9 |
| ST output (MW) | 213 | 212.1 | 210.5 | 209.79 | 209.1 | 207.5 | 205.9 |
| Gross output (MW) | 554 | 573.65 | 591.44 | 600.02 | 609.18 | 626.3 | 643.98 |
| Total Auxiliary (MW) | 153.5 | 153.9 | 154.4 | 154.6 | 154.8 | 155.2 | 155.6 |
| Net output (MW) | 401.8 | 419.6 | 437.0 | 445.4 | 454.3 | 471.1 | 488.3 |
| Gross efficiency % | 44.11 | 44.84 | 45.54 | 45.88 | 46.22 | 46.88 | 47.52 |
| Net efficiency % | 31.92 | 32.80 | 33.65 | 34.06 | 34.47 | 35.26 | 36.03 |
| Description | BC | C-25 | C-26 | C-27 | C-28 | C-29 | C-30 |
| X_{air} | 0 | 50 | 50 | 50 | 50 | 50 | 50 |
| X_{N_2} | 0 | 20 | 40 | 50 | 60 | 80 | 100 |
| Extracted Air kg/sec | 0 | 83.57 | 83.57 | 83.57 | 83.57 | 83.57 | 83.57 |
| Air from the compressor kg/sec | 685 | 685 | 685 | 685 | 685 | 685 | 685 |
| Syngas flow kg/sec | 30.09 | 28.37 | 28.74 | 29.13 | 29.52 | 29.91 | 30.28 |
| Exhaust gas inlet temperature GT °C | 1430 | 1430.9 | 1430.1 | 1430.4 | 1430.7 | 1430.5 | 1430.4 |
| Coal thermal energy LHV MW _{th} | 1258 | 1157.7 | 1179.5 | 1202.3 | 1225.1 | 1247.4 | 1269.9 |
| GT output | 341.6 | 272.9 | 292.1 | 311.7 | 331.4 | 350.8 | 370.3 |
| ST output | 213.3 | 220.9 | 219.3 | 217.7 | 216.1 | 214.4 | 212.6 |
| Gross output | 554.9 | 493.8 | 511.4 | 529.5 | 547.5 | 565.2 | 583.0 |
| Total Auxiliary MW | 153.5 | 100.5 | 101.0 | 101.5 | 102.06 | 102.55 | 103.5 |
| Net output | 401.8 | 393.3 | 410.4 | 427.9 | 445.4 | 462.7 | 479.9 |
| Gross efficiency % | 44.11 | 42.65 | 43.36 | 44.03 | 44.69 | 45.31 | 45.90 |
| Net efficiency % | 31.92 | 33.97 | 34.79 | 35.59 | 36.36 | 37.09 | 37.79 |
| Description | BC | C-31 | C-32 | C-33 | C-34 | C-35 | C-36 |
| X_{air} | 0 | 95 | 95 | 95 | 95 | 95 | 95 |
| X_{N_2} | 0 | 20 | 40 | 50 | 60 | 80 | 100 |
| Extracted Air kg/sec | 0 | 160.7 | 160.7 | 160.7 | 160.7 | 160.7 | 160.7 |
| Air from the compressor kg/sec | 685 | 685 | 685 | 685 | 685 | 685 | 685 |
| Syngas flow kg/sec | 30.09 | 26.76 | 27.25 | 27.72 | 28.20 | 28.68 | 29.16 |
| Exhaust gas inlet temperature GT °C | 1430.3 | 1430.8 | 1431.1 | 1430.5 | 1430.3 | 1430.1 | 1430.4 |
| Coal thermal energy LHV MW _{th} | 1258 | 1063.8 | 1092.2 | 1119.8 | 1147.7 | 1175.6 | 1204.1 |
| GT output MWe | 341.8 | 209.1 | 229.1 | 248.7 | 268.4 | 288.2 | 308.2 |
| ST output MWe | 213.5 | 227.7 | 225.6 | 223.6 | 221.5 | 219.5 | 217.4 |
| Gross output MWe | 555.3 | 436.8 | 454.7 | 472.3 | 490.1 | 507.7 | 525.6 |
| Total Auxiliary MWe | 153.5 | 31.54 | 32.17 | 32.78 | 33.396 | 34.01 | 34.63 |
| Net output | 401.8 | 405.3 | 422.5 | 439.54 | 456.63 | 473.72 | 491.2 |
| Gross efficiency % | 44.11 | 41.06 | 41.63 | 42.17 | 42.69 | 43.18 | 43.65 |
| Net efficiency % | 31.92 | 38.09 | 38.68 | 39.24 | 39.78 | 40.29 | 40.78 |

V. RESULTS AND DISCUSSION

A. Study on air extraction rate X_{air} without, partial and full injection of N_2

Table 3 illustrates the effects of air extraction rate at conditions of no N_2 infusion, partial 50% injection, and complete 100% injection. For the first case, no N_2 injection was considered where the remarkable drop is observed in E_{GT} output due to air extraction from 0 to 100% decreased to the combustor. The air extraction causes an increase in temperature, in order to maintain TIT temperature, the flow of syngas regulated. Thus, creating a significant loss to GT output and a firm agreement to Han et al. [32] and Cormos et al. [35]. However, it observed that the production of ST kept on increasing as the air extraction flow increases. Although the mass flow rate across the GT decreased, the sensible heat and TIT of GT remained constant due to which the ST output slightly increased. The extraction of air minimized the load consumption on ASU, but the overall performance of the plant has significantly affected by this technique. The air

extraction from the GT compressor observed with almost the same trend at partial 50% induction of N_2 to GT combustor. As the mass flow rate of air extracted from 0% to 100% increases; the power output by GT decreases due to the reduction of overall flow across the GT. At 0% and 100% X_{air} , the power produced by GT was 390.2 MW and 258.3 MW and by ST was 209.7 MW and 222.5 respectively. However, the stability in TIT and sensible heat of exhaust gas causes a slight increase in the performance of ST, which contradicts from the study in the literature [32]. Similarly, the syngas flow is also regulated with TIT restriction. For cases between, C13-C18, the same trend was observed in required indicators at full injection of N_2 to GT combustor with air extraction growth from 0% to 100% but the power augmented effect for this case is remarkably apparent due to the large flow of exhaust at 0% extraction of air but goes on decreasing by increasing air extraction flow. The full injection of N_2 causes the maximum output of GT. For each case, the flow across the GT draws significant loss to the power output with X_{air} growth.

B. Study on N₂ injection at without, partial and full load

Table 4 shows the injection rate of N₂ from 0% to 100% to GT combustor against without, a partial and complete amount of air extracted from the primary air compressor (PAC) of GT. In the case C19-C24, the N₂ injection rate studied against without air extraction from the primary air compressor or in other words the ASU operated at its full capacity. It is seen, as the N₂ induction increases, the output power across GT increases. Conversely, the ST power output decreases as the sensible heat decreases with the introduction of N₂ injection. The elemental N₂ causes a drop in temperature during the burning of syngas inside the combustion chamber [5]. In most cases, the temperature of the flame is regulated against by nitrogen. Similarly, for cases, C25-C30, where the ASU unit is running at 50% (50% extraction of air from PAC), the same trend in the power output of GT was substantially reduced as compared to Case C4. Although, the power output increased with nitrogen induction; however, the overall flow across the GT has diminished. In the final case study of case C6, the nitrogen induction rate was observed by operating ASU at full load, i.e. 100% extraction from PAC. The overall output produced by GT and ST reduced due to the reduction of the flow of fresh air from the PAC. The syngas flow rate was regulated to meet the temperature limitation for GT, and in Table 4, the syngas flow rate increases as the air extraction flow increases.

VI. CONCLUSION

The purpose of the study is to evaluate the ASU-GT integration by considering the air extraction from the PAC and nitrogen induction to GT combustor. The whole research comes up with the following main points:

1. The study explains the importance of air extraction that caused a significant power production on GT as well as ST. The air extraction rate plays a vital role in process optimization if the ASU is operating at elevated pressure.
2. The air extraction causes a remarkable change in the performance of GT as compared to nitrogen induction, but the combination of both can lead to optimized efficiency.
3. In Table 3, the syngas flow rate decreased to meet TIT limitations against the increase in air extraction rate. On the Contrary, in Table 4, the syngas flow increased because of the induction of N₂ causes the mitigation of combustion temperature of exhaust gas.
4. By comparing with the base case, four optimized solutions were obtained by ASU-GT integration, i.e., C17, C18, C34, and C35, where the gross efficiency and net efficiency is remarkably high.

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